Chapter 3 POWERHOUSE

Chapter 3 Powerhouse

This chapter is prepared to supplement planning of the powerhouse of the Jatibalang Multipurpose Dam in terms of civil engineering design.

3.1. Scope of Design

The powerhouse dealt within this chapter includes the structures from some part of the penstock to the end of the tailrace. The following structures, therefore, are designed and presented in this chapter.

(1) Penstock : The penstock from just 2 m upstream of the powerhouse to the inlet

valve

(2) Powerhouse: The structure below EL. 98.00 excluding architectural works above

EL. 97.50

(3) Tailrace : The draft pit and the tailrace culvert under the powerhouse is

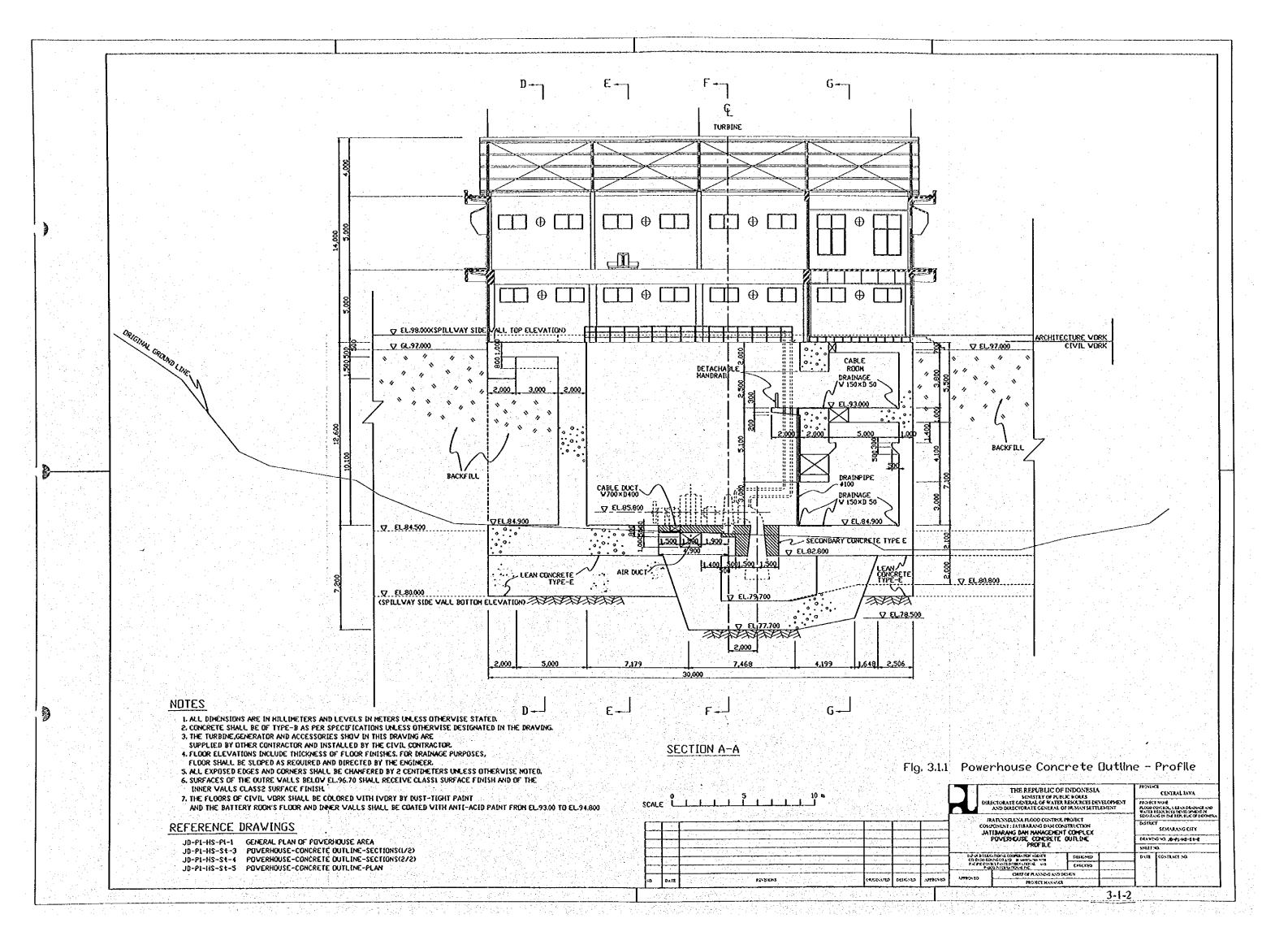
designed and the reinforcement for the tailrace passing through the

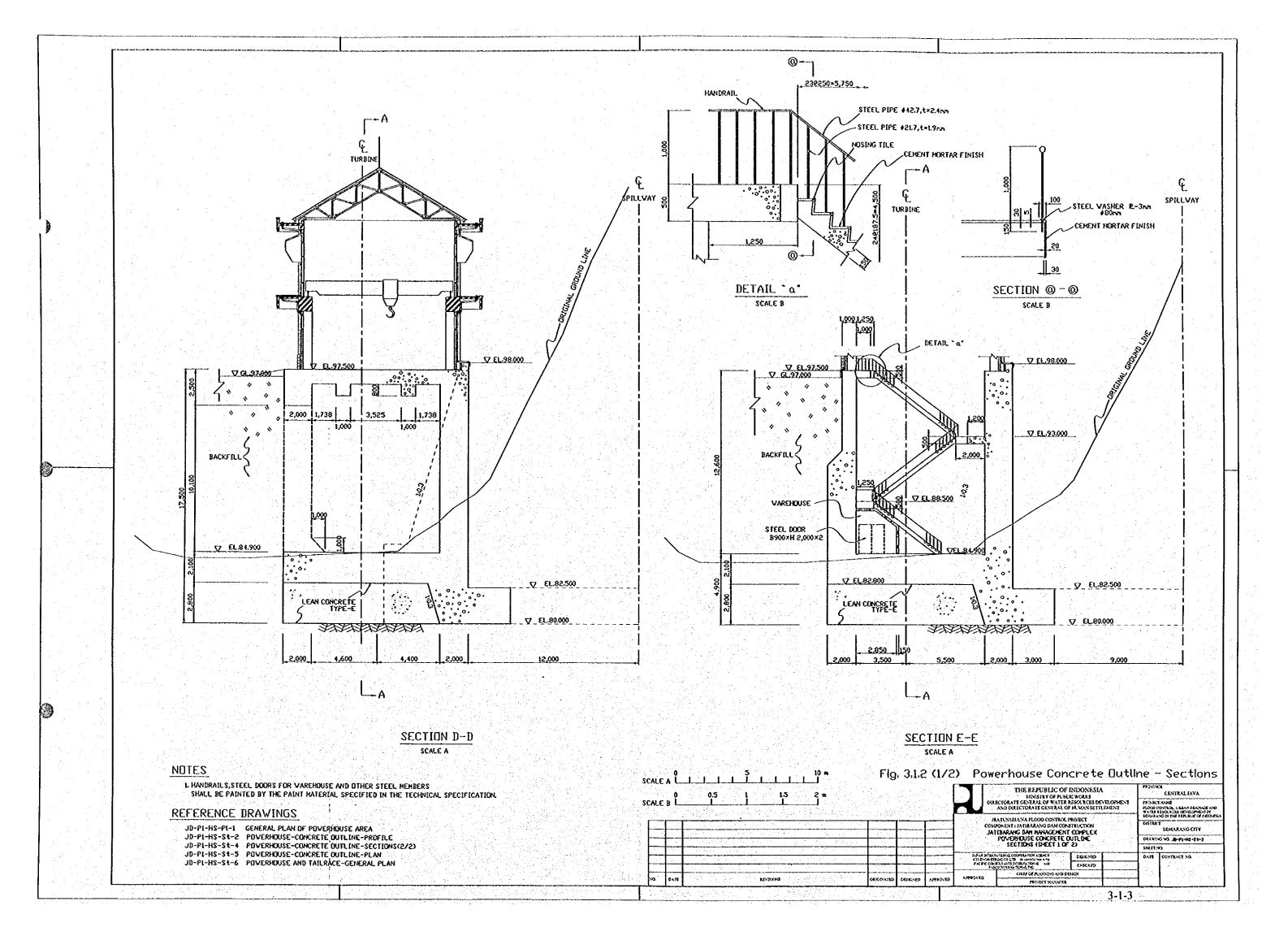
foot of spillway was studied.

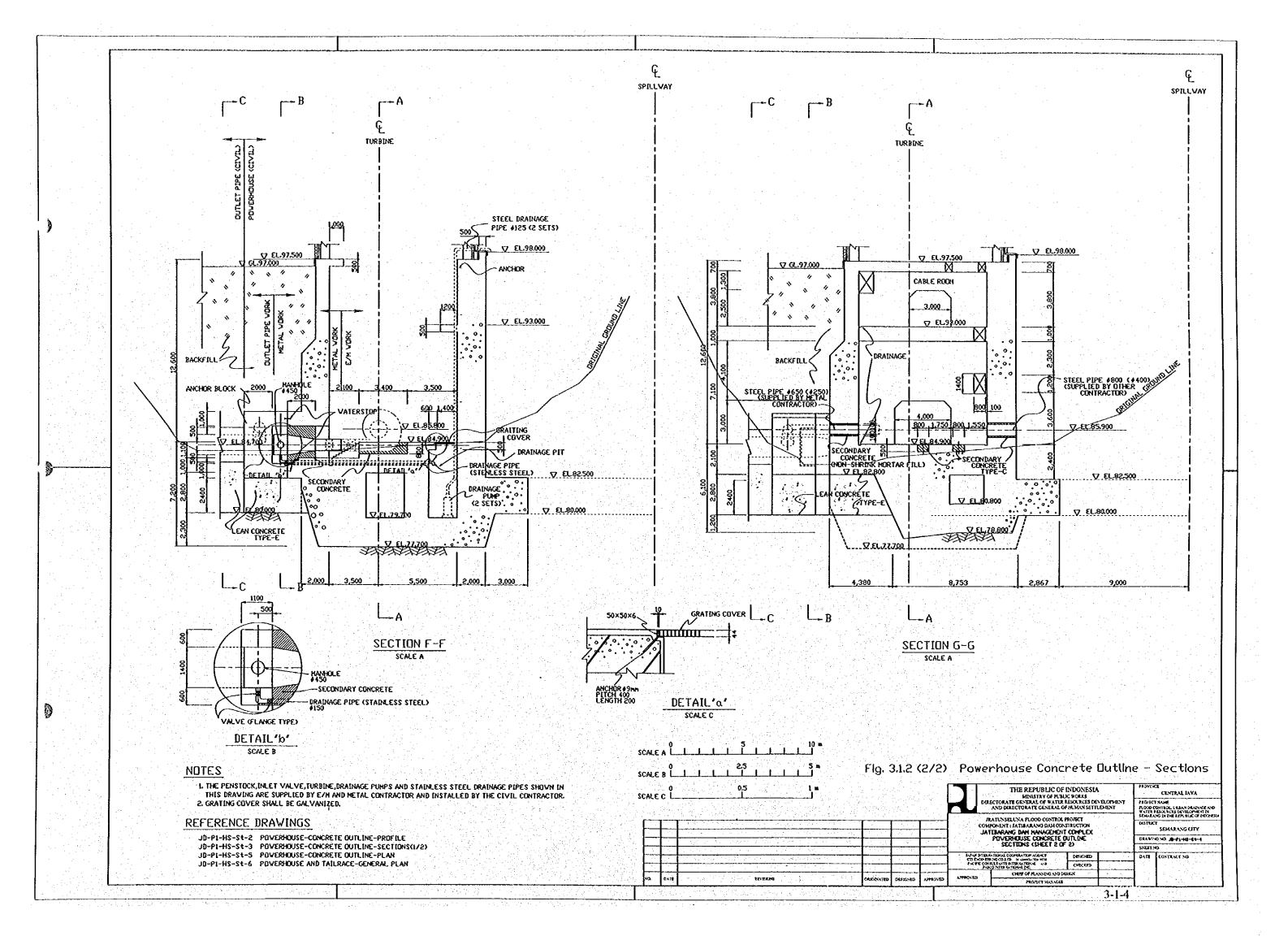
(4) Tailrace Gate

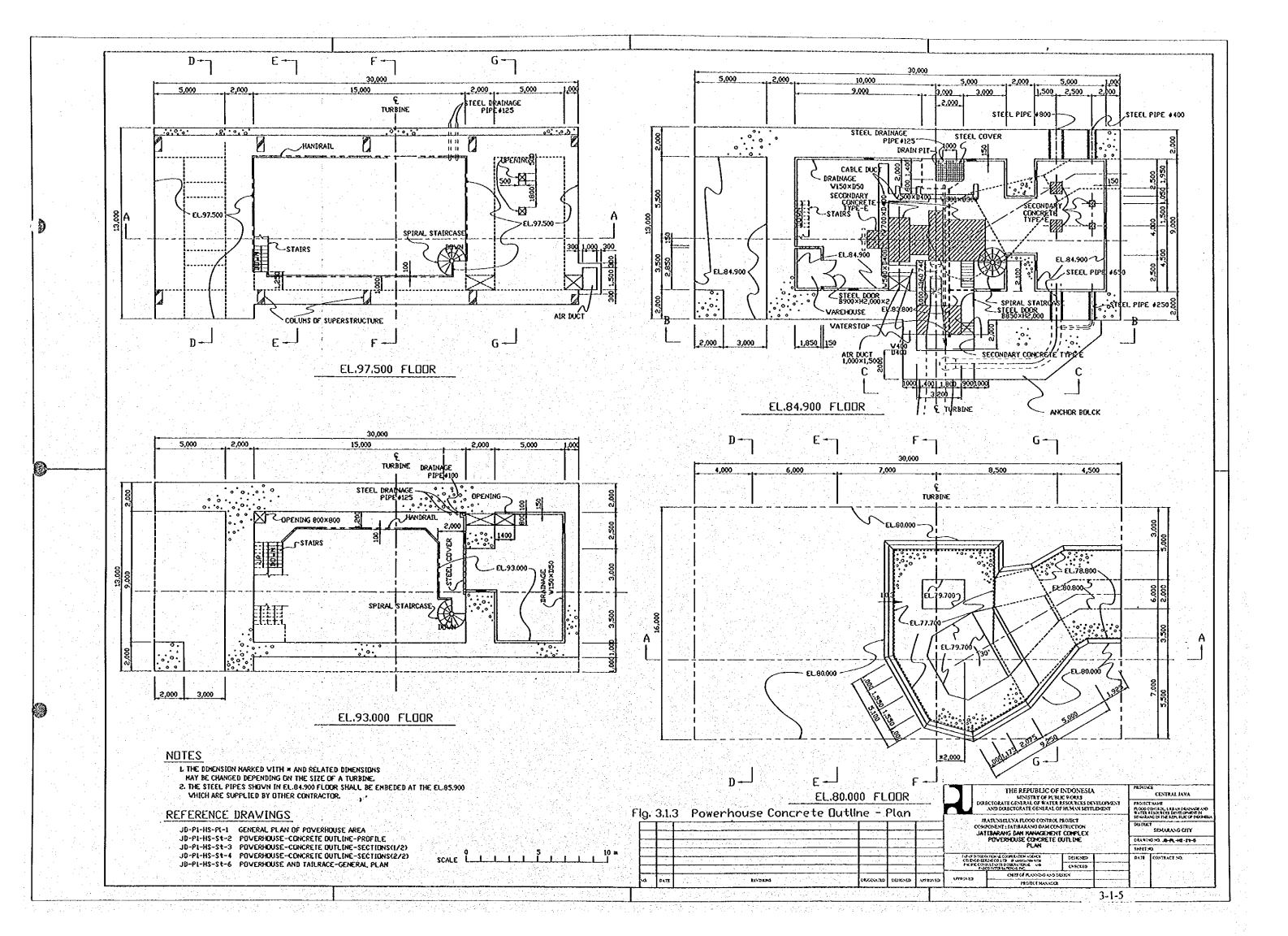
For the design of the above structures, water hammer analysis was carried out. The head loss was also calculated to seek the installed capacity and energy to be produced by the powerhouse.

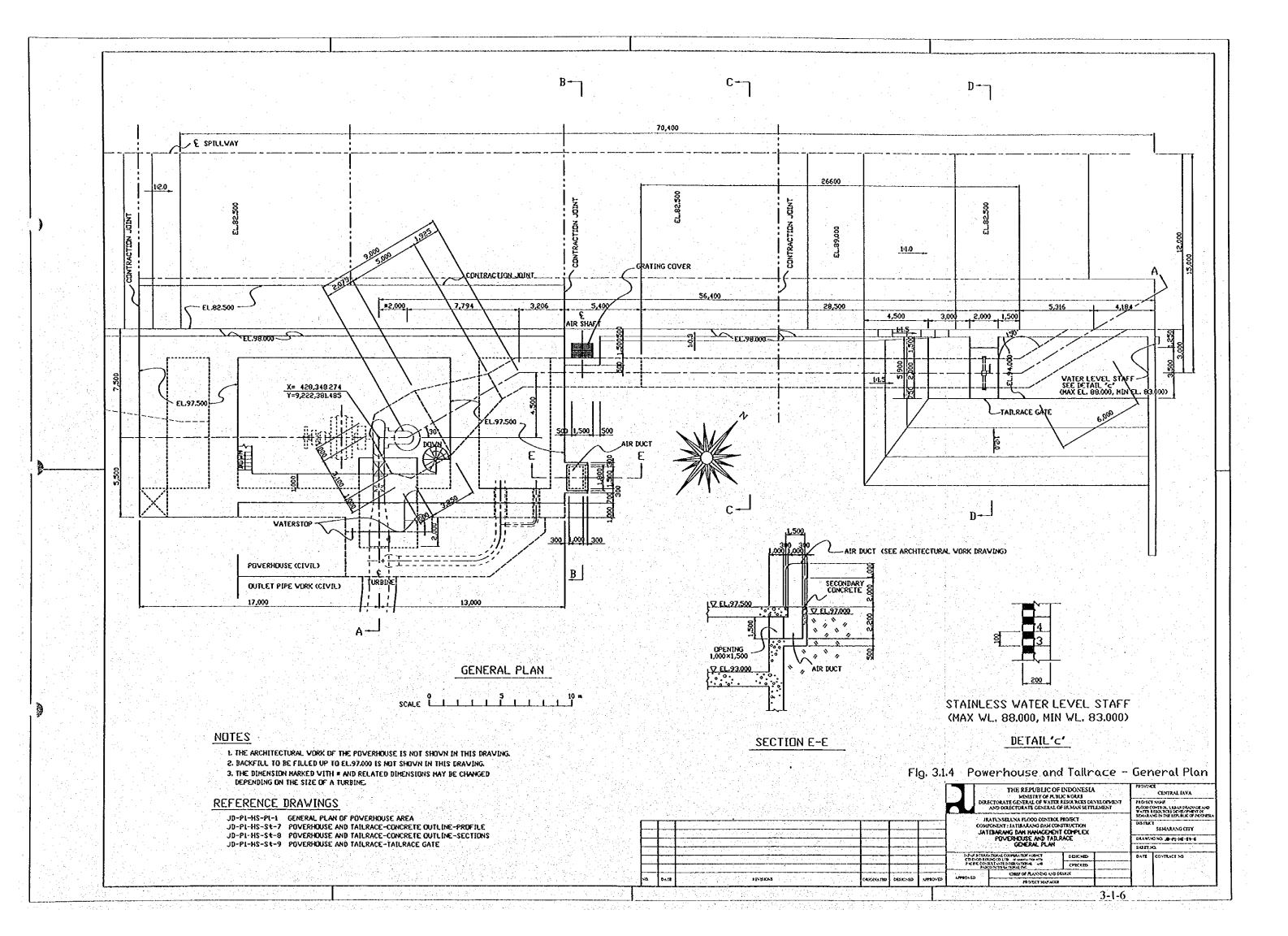
The concrete outline and reinforcement of the structures are shown in Figs.3.1.1 to 3.1.13.

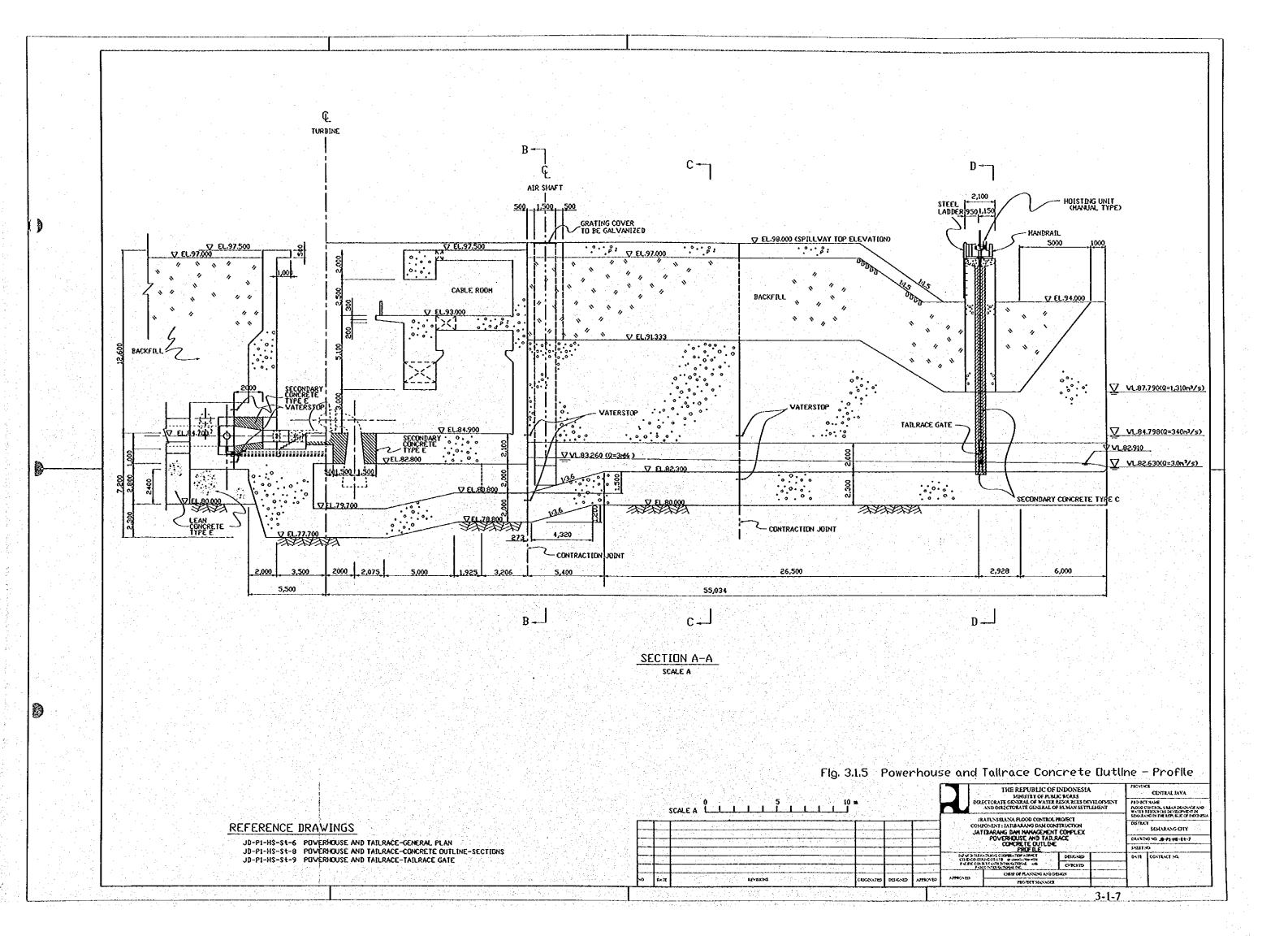


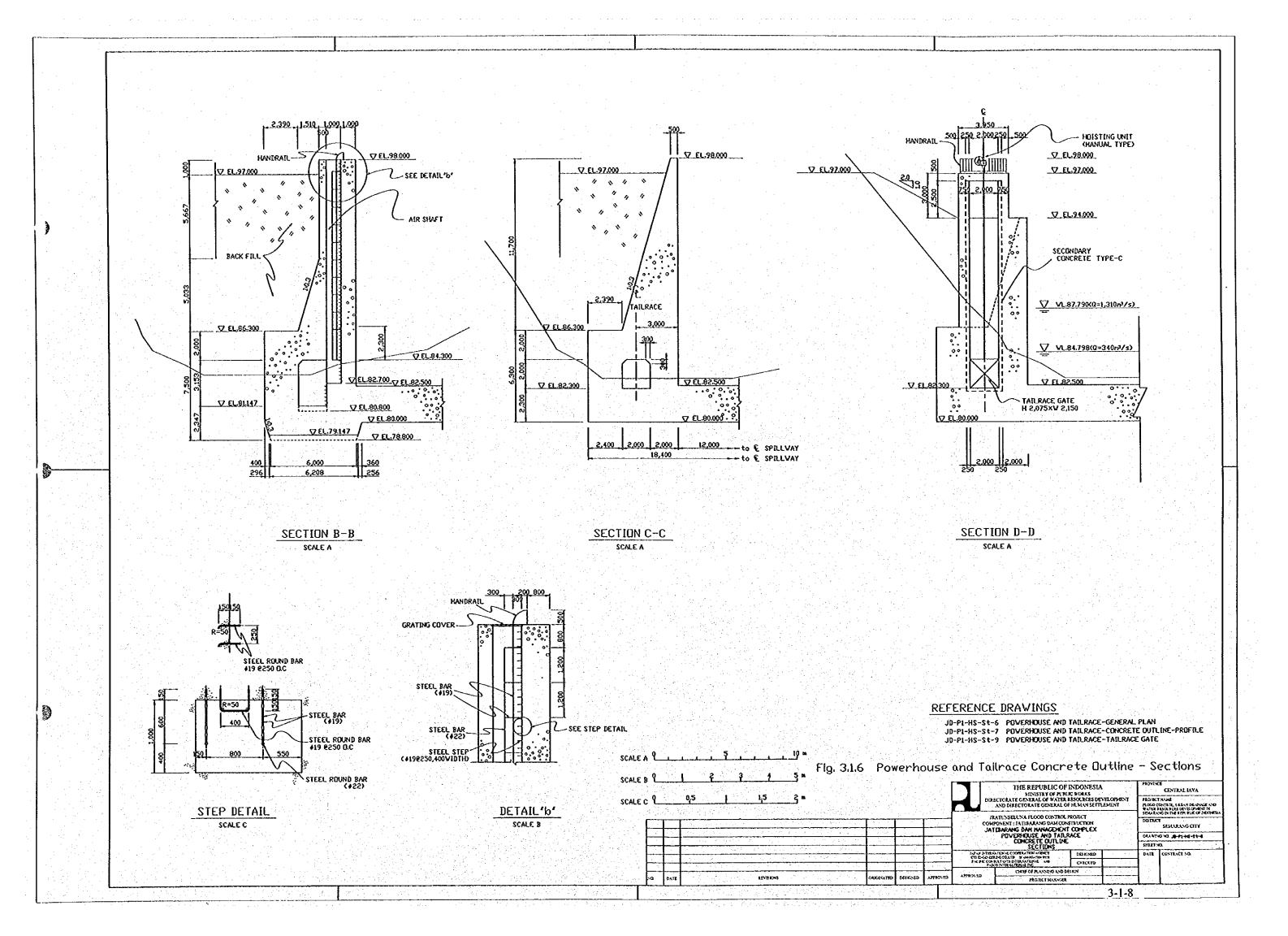












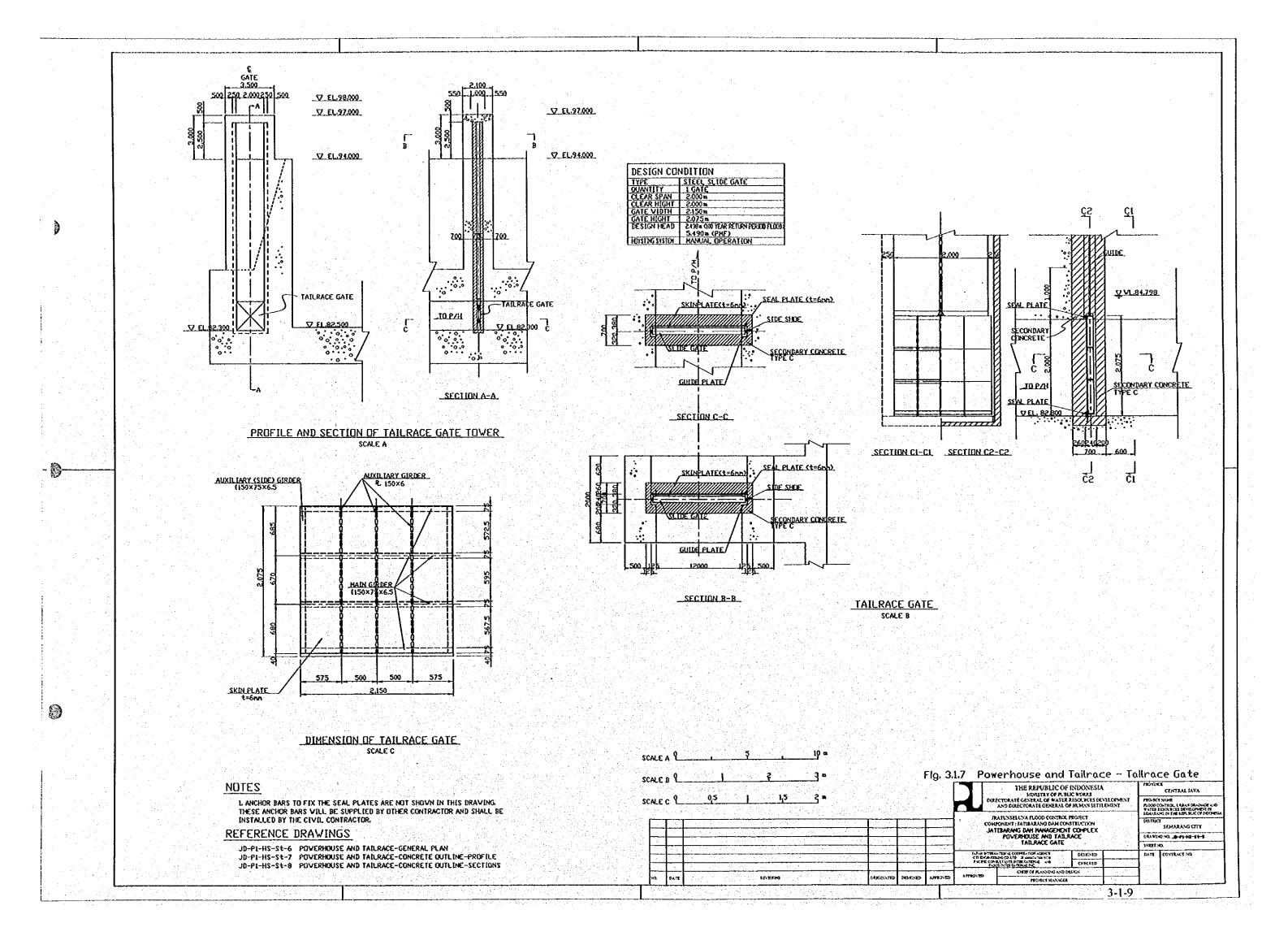


Fig. 3.1.8 Powerhouse Reinforcement Section (1/4)

Fig. 3.1.9 Powerhouse Reinforcement Section (2/4)

Fig. 3.1.10 Powerhouse Reinforcement Section (3/4)

Fig. 3.1.11 Powerhouse Reinforcement Section (4/4)

Fig. 3.1.12 Powerhouse Reinforcement Plan (1/2)

Fig. 3.1.13 Powerhouse Reinforcement Plan (2/2)

3.2. Main Features

Main features of the powerhouse are tabulated in the following table.

	Description	Unit	Main Feature
(1)	Hydropower Generation		
٠	- Maximum plant discharge	m³/s	3.00
	- Maximum gross head	m	69.10
	- Design head	m	64.38
	- Installed capacity	kW	1,550
(2)	Dam and Water Level		
	- Reservoir NWL	EL. m	148.90
	- Reservoir LWL	EL. m	136.00
	- Tail water level TWL	EL. m	82.91

3.3. Geological Condition in the Powerhouse Area

The powerhouse will be founded on the same rock formation as that of the spillway stilling basin. Judging from the geological profile made along the centerline of the spillway and outlet facilities, rock foundation for the powerhouse would be expected at EL. 80.80.

3.4 Effective Head

3.4.1. Reservoir Water Level and Tailrace Water Level

1) Reservoir Water Level

The normal water level (NWL) of EL.148.9m was used for calculating the effective head and annual generation.

2) Tailrace Water Level

The water level at the tailrace exit is defined as the tailrace water level (TWL) for generation. Generally, TWL is equal to the water level of the downstream river channel when it is higher than the water level at the tailrace exit. The water level of the downstream river channel can be obtained by the H-Q curve of the river reach downstream of the spillway shown in Table 3.4.1 and Fig.3.4.1. The water level of the downstream river channel and depth at the outlet section of the tailrace corresponding to the maximum plant discharge of Q= 3 m³/s are as follows:

Discharge	Water Level (m)	Water Depth (m)
Maximum plant discharge (Q=3 m³/s)	82.631	0.331

Notes: The bottom elevation of the tailrace at exit is 82.3m.

On the other hand, the critical depth at the outlet section is calculated by

$$h_c = [\alpha Q^2/(gB)]^{1/3}$$

Where

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h_c: critical depth (m)

Q: discharge (m³/s)

B: width of section (m)

α: kinetic energy correction factor

g: gravity acceleration

The calculated parameters are given

$$\alpha = 1.0$$

 $O = 3.0 \text{ m}^3/\text{s}$

B = 2.0 m

Therefore, the critical depth is equal to

$$h_c = [\alpha Q^2/(gB)]^{1/3} = (1.0 \times 3^2/9.8/2)^{1/3} = 0.612 \text{ (m)}$$

It can be found that the critical depth is larger than the water depth of the downstream river channel of 0.331m. Thus, the water level of the downstream river channel has no effect on the water depth in the tailrace, consequently, the water level at the tailrace exit is determined by the bottom elevation and critical depth. Hence, the water level at the tailrace exit is obtained by

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$$H = 82.3 + h_c = 82.3 + 0.612 = 82.912 \text{ m}$$

Therefore, the TWL for calculating effective head and generation is taken as EL.82.91m.

3.4.2. Computation of Head Loss

1) Design Condition

The Jatibarang Power Station is designed as a part of the multipurpose reservoir. The maximum power discharge of 3 m³/s is conveyed by a branched pipe which is a part of the outlet facilities that consist of an inclined intake structure with bulkhead and emergency gate, outlet tunnel, steel outlet pipe, control gate, penstock, tailrace gallery and tailrace gate. The bifurcation valve is set just before the power station so as to enable to directly release the water for domestic use without passing the power station. The maximum design discharge of the outlet pipe is 6 m³/s, which is used for determining the size of the outlet facilities before the bifurcation. The maximum power discharge of 3 m³/s is used for computing the head losses of the whole outlet facilities for hydropower

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generation. The head loss computation is carried out by dividing the outlet facilities into three portions, that is,

- a) Intake structure including trash racks
- b) Outlet steel pipe
- c) Tailrace

The schematic of the outlet facilities is shown in Fig. 3.4.2.

2) Head Loss Calculation Formulae

The calculation of the head losses was principally carried out in accordance with the formulas and hydraulic coefficients as stipulated in the Design Criteria Report, Vol.2, CTI Engineering Co., Ltd., Pacific Consultants International and PASCO International Inc., March 1999.

The roughness coefficients for different waterways are selected as follows

Concrete n = 0.0125Steel n = 0.0115

The value of the gravity acceleration is taken as 9.8m/s². Other coefficients used for the loss calculation are explained in respective section.

3) Head Loss of Intake Portion

The head losses of the intake portion is consisted of the head loss through trash racks, friction loss and entrance loss.

(1) Head Loss through Trash Racks

The head loss through the trash racks is computed by

$$H_i = \alpha \cdot K_i \cdot \frac{V_i^2}{2g} = \alpha \cdot K_i \cdot \frac{1}{A_i^2} \cdot \frac{Q^2}{2g}$$

$$K_1 = \beta \cdot \sin \theta \cdot \left(\frac{t}{b}\right)^{1/3}$$

Where

H_i: head loss through trash racks (m)

K_t: loss coefficient through trash racks

V₁: mean velocity before trash racks (m/s)

 α : safety factor for clogging due to trash

 β : bar shape coefficient

0: trash rack inclination (°)

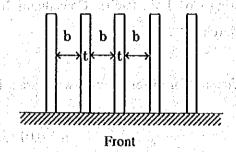
t: width of bar (m)

b: clear span between bars (m)

Q: discharge (m³/s)

A₁: flow sectional area before trash racks (m²)

g: gravity acceleration



Trash Rack

$$\beta = 1.60$$

$$\beta = 1.77$$

$$\beta = 2.34$$

$$\beta = 1.73$$

$$\beta = 0.73$$

Bar Shape Coefficient

 $\beta = 2.34$ $\theta = \tan^{-1}(1:1.4)=35.538^{\circ}$ t/b = 0.25 $\alpha = 3.0$

 $Q = 3 \text{ m}^3/\text{s}$

Since

 $A_1 = 2.0 \times 18.9 = 37.8 \text{ m}^2$

Therefore, the head loss through trash rack is equal to

$$K_t = 2.34 \times \sin(35.538) \times 0.25^{4/3} = 0.214$$

 $H_t = 3.0 \times 0.214 \times (3/37.8)^2 / 19.6 = 0.0002 = 0.000 \text{ (m)}$

(2) Friction Head Loss of Intake Structure

The following formula is used for the calculation of the friction loss of the intake structure.

$$H_f = K_f \cdot L/R \cdot V^2/(2g) = K_f \cdot L/R \cdot (Q/A)^2/(2g)$$

 $K_f = 2g \cdot n^2/R^{1/3}$

Where

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H_f: friction head loss (m)

K_t: friction coefficient

n: roughness coefficient

L: length of pipe (m)

V: flow velocity (m/s)

Q: discharge (m³/s)

R: hydraulic radius = A/S

A: flow area (m²)

S: wetted perimeter (m)

g: gravity acceleration

R = A/S = 3.2/7.2 = 0.444 (m)

The following calculation conditions are used.

2.0m

Therefore, the friction head loss of the intake structure is

$$K_f = 19.6 \times 0.0125^2 / 0.444^{1/3} = 0.004$$

 $H_f = 0.004 \times 24.939 / 0.444 \times (3/3.2)^2 / 19.6 = 0.010 \text{ m}$

(3) Entrance Loss

The entrance loss from the intake structure into the outlet pipe is calculated by

$$H_e = K_e \cdot V^2 / (2g) = K_e \cdot (Q/A)^2 / (2g)$$

Where

H_e: entrance loss (m)

K.: entrance loss coefficient

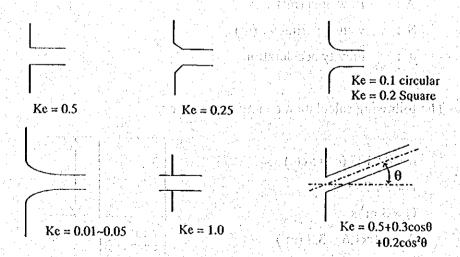
V: flow velocity (m/s)

Q: discharge (m³/s)

A: flow area at entrance (m²)

g: gravity acceleration

The values of Ke are as follows:



Entrance Loss Coefficient

The discharge and flow area are

$$Q = 3 \text{ m}^3/\text{s}$$

$$A = 1.4 \times 1.4 = 1.96 \text{ m}^2$$

From the above figure, the entrance loss coefficient is selected as $K_e = 0.05$. Therefore, the entrance loss is computed to be

$$H_c = 0.05 \times (3/1.96)^2 / 19.6 = 0.006 \text{ m}$$

(4) Summary of Head Loss of Intake Portion

Assuming the allowance of 1% for the above-mentioned calculated loss, the head loss of the intake structure is summarized as follows

Through trash rack: 0.000 m

Friction: 0.010 m

Entrance: 0.006 m

Allowance (1%): 0.000 m

Total: 0.016 m

4) Head Loss of Steel Pipe

The head losses of the steel pipe include the friction loss, gradual contraction loss, bending loss and the loss due to bifurcation.

(1) Friction Head Losses of Steel Pipe

The friction losses are composed of the loss through the portions with the same diameter and the one through the gradual contraction portion. The different formulae are employed for calculating the losses through the different portions.

A. Losses through the portions with the same diameter ...

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The friction losses of the portions with the same diameter are computed by

$$H_1 = 124.5n^2/D^{4/3}LV^2/(2g) = 124.5n^2/D^{4/3}L(Q/A)^2/(2g)$$

Where

H₁: friction head loss (m)

n: roughness coefficient

D: internal diameter of pipe (m)

L: length of pipe (m)

V: flow velocity (m/s)

Q: discharge (m³/s)

A: flow area (m²)

g: gravity acceleration

The calculated parameters and friction losses of the portions with the same diameter are summarized in the following table.

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Friction Losses of portions with same diameter

Part	D (m)	n	L(m)	A (m ²)	H ₁ (m)
1(before branching)	1.4	0.0115	403.161	1.539	0.822
2(just after branching)	1.4	0.0115	2.000	1,539	0.004
3(after gradual portion)	0.8	0.0115	1.500	0.503	0.060

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B. Losses through Gradual Contraction Portion

The friction loss through the gradual contraction portion is calculated by

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$$H_1 = 2.37n^2LQ^2(1/D_2^{13/3}-1/D_1^{13/3})/(D_1-D_2)$$

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H₁: friction head loss (m)

n: roughness coefficient

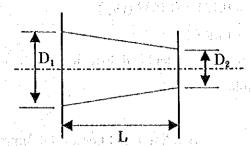
D₁: internal diameter before contraction (m) and

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D₂: internal diameter after contraction (m)

L: length of pipe (m)

Q: discharge (m³/s)



The parameters and calculated friction loss through the gradual contraction portion are summarized in the following table.

Friction Losses through Gradual Contraction Portion

Part	D _I (m)	D ₂ (m)	n	L (m)	H _t (m)
1 (D1400 to D800)	1.400	0.800	0.0115	2.860	0.032

(2) Bending Head Losses

The horizontal and vertical friction losses between the intake structure and the branching valve are computed in this study. The location of the bends is shown in Fig.3.4.4. The following formula is employed for calculating the bending losses.

$$H_b = K_{b1} \cdot K_{b2} \cdot V^2 / (2g) = K_{b1} \cdot K_{b2} \cdot (Q/A)^2 / (2g)$$

Where

H_b: bending head loss (m)

K_{b1}: loss coefficient determined by the ratio (=ρ/D) of the bending radius ρ to the pipe diameter D, in case that a center angle of bending is 90°{Anderson-Straub adjustment value is given in Figure (a)}

i patient to

 K_{b2} : ratio of the loss for a center angle θ to the loss for a center angle of 90°

V : flow velocity (m/s)

Q : discharge (m³/s)

A: flow area (m²)

g : gravity acceleration a separated by

The Anderson-Straub adjustment values of K_{b1} and K_{b2} are defined empirically as follows

$$K_{b1} = 0.131 + 0.1632 (D/\rho)^{7/2}$$

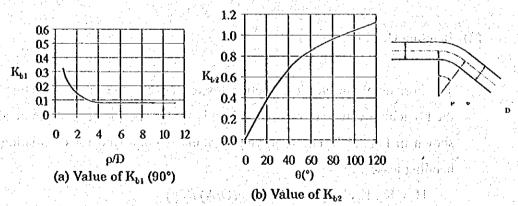
$$K_{b2} = (0/90^{\circ})^{1/2}$$

The parameters and calculated loss at each bend are summarized in the following table.

Bending Head Losses (Q=3m³/s)

	No. of	D			1 13			K ₈₁ K ₈₂	
	Bend	(m)	ρ (m)	θ (°)	K _{b1}	K _{b2}	A (m ²)	$/A^2$	H _b (m)
	VIP1	1.400	2.800	50°33'27"	0.145	0.749	1.539	0,046	0.021
	HJP1	1.400	45.000	90°	0.131	1.000	1.539	0.055	0.025
ľ	HIP2	1.400	45.000	45°	0.131	0.707	1.539	0.039	0.018
	VIP2	1.400	4.200	3°54'18"	0.134	0.208	1.539	5 , 0.012	0.006
	HIP3	1.400	4.200	19°21'04"	0.134	0.464	1.539	0.026	0.012
	Total	1.55%				UM II		1 895. 4 July 1	0.082

Notes: The values of K_{b1} and K_{b2} are obtained by the empirical formulae.



Bending Loss Coefficient

(3) Branching Loss

The following Gardel formula is used for calculating the head loss due to branching.

$$H_b = H_\alpha - H_\beta = K_b \cdot V_\alpha^2 / (2g) = K_b \cdot (Q_\alpha / A_\alpha)^2 / (2g)$$

 $K_b = 0.58 q_\beta^2 - 0.26 q_\beta + 0.03$

Where

H_b: head loss due to branching

K_b: branching head loss coefficient

 H_{α} : working pressure head before branching (m)

H, : working pressure head after branching (m)

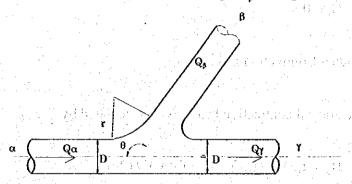
V_α: flow velocity before branching (m/s)

Q_α: original discharge (m³/s)

 A_{α} : flow area before branching (m²)

D: internal diameter of pipe (m)

 q_{β} : ratio of branched discharge Q_{β} to original discharge Q_{α}



The calculated parameters are

$$Q_{\alpha} = 3.0 \text{ m}^3/\text{s}$$
 $Q_{\beta} = 0 \text{ m}^3/\text{s}$
 $q_{\beta} = Q_{\beta}/Q_{\alpha} = 0$
 $D=1.4 \text{ m}$
 $A_{\alpha} = \pi D^2/4 = 3.1415926 \times 1.4^2/4 = 1.539 \text{ (m}^2)$

Therefore, the branching head loss becomes as follows:

$$K_b = 0.58q_{\beta}^2 - 0.26q_{\beta} + 0.03 = 0.03$$

 $H_b = K_b(Q_o/A_o)^2/(2g) = 0.03 \times (3/1.539)^2/19.6 = 0.006 \text{ m}$

(4) Valve Loss

The head loss due to valve is calculated by

$$H_v = f_v \cdot V^2 / (2g) = f_v \cdot (Q/A)^2 / (2g)$$

Where

H_v: head loss due to valve

 f_v : valve head loss coefficient, $f_v = 0$, when the valve is full open

V: flow velocity (m/s)

Q: discharge (m³/s)

A: flow area of pipe (m²)

D: internal diameter of pipe (m)

Since f_v= 0 is applicable to calculating the effective head, then

 $H_v = 0$

(5) Gradual Contraction Loss

The gradual contraction head loss is calculated by

$$H_{gc} = f_{gc} \cdot V_2^2 / (2g) = K_{gc} \cdot (Q/A_2)^2 / (2g)$$

Where

H_{sc}: gradual contraction loss (m)

f_{gc}: gradual contraction loss coefficient, it is a function of the contraction angle 0 and the ratio of the cross sectional areas of the flow before and after contraction, A₂/A₁(see the following figure)

V₂: velocity after contraction (m/s)

Q : discharge (m³/s)

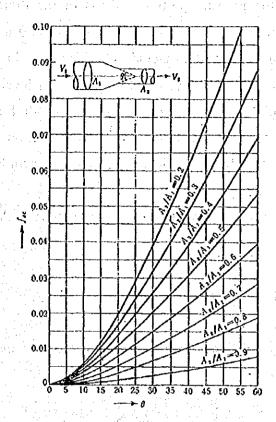
A₁: sectional flow area before contraction (m²)

A₂: sectional flow area after contraction (m²)

The parameters of the gradual contraction portion and calculated gradual contraction loss are given in the following table.

Gradual Contraction Loss

D (mm)	A ₁ (m ²)	A ₂ (m ²)	A ₂ /A ₁	0 (°)	f _{gc}	f_{gc}/A_2^2 (m)	h _{gc} (m)
1400 to 800	1.539	0.503	0.327	11°58'34"	0.006	0.0237	0.011



Contraction Loss Coefficient

(6) Summary of Head Loss through Steel Pipe

Assuming the allowance of 1% for the above-mentioned calculated loss, the head loss through the steel pipe is summarized as follows

Friction:	0.918 m
Gradual contraction:	0.011 m
Bending:	0.082 m
Bifurcation:	0.006 m
Valve:	0.000 m
Allowance:	0.010 m
Total:	1.027 m

5) Head Loss of Tailrace

The head losses of the tailrace consist of the head loss of the pressure portion (upper part of the tailrace) and non-pressure portion (lower part of the tailrace).

The head loss of the pressure portion consists of the draft-tube outlet loss, bending loss, sudden contraction and expansion losses and friction loss. The head loss along the non-pressure portion includes the friction loss and exit loss due to velocity decrease. The exit loss by abrupt drop of water level is not considered because the critical depth is selected as the exit depth for determining the effective head. The bending loss along non-pressure portion is also negligible. The typical sections for calculating the tailrace loss are shown in Fig. 3.4.3.

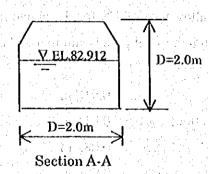
- (1) Head Loss along Non-pressure Portion (between Section A-A and Section C-C)
 - A. Velocity Head Loss at Exit

From the calculation in the above section, the flow area at the exit of the tailrace is

$$A = Bh_c = 2.0 \times 0.612 = 1.224 \text{ m}^2$$

Therefore, the loss at the exit due to velocity decrease is

$$H_{\text{exit}} = V^2/(2g) = (Q/A)^2/(2g) = 1.0 \times (3/1.224)^2/19.6 = 0.307 \text{ (m)}$$



B. Friction Loss along Non-pressure Portion

The friction loss along the non-pressure portion is calculated by

$$H_{Lf} = \Delta L \cdot \overline{S_f}$$

$$\Delta L = \Delta E/(i - \overline{S_f}) = (E_2 - E_1)/(i - \overline{S_f})$$

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$$\overline{S_f} = 0.5 \left(\frac{1}{R_1^{4/3}} \cdot \frac{n^2 V_1^2}{2g} + \frac{1}{R_2^{4/3}} \cdot \frac{n^2 V_2^2}{2g} \right)$$

$$E = h + \alpha \cdot \frac{V^2}{2g}, \qquad i = (z_1 - z_2) / \Delta L$$

Where

h_{1f}: friction loss (m)

E: specific energy (m)

 ΔE : change in specific energy (= $E_2 - E_1$)

h : water depth (m)

 $\overline{S_I}$: average friction slope

V: velocity of flow (=Q/A)

Q : discharge

A : flow area of an electronic or positi

R : hydraulic radius Hard France

i way: bottom slope was the applied

z : bottom elevation

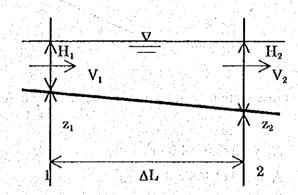
ΔL: interval distance

n : roughness coefficient

g : gravity acceleration

 α : kinetic energy correction factor (=1.0)

Suffixes 1 and 2 indicate the upstream and downstream sides of the reach of length, ΔL .



The upstream depth h_1 is calculated by direct step method. The calculated conditions and results are shown in Table 3.4.2. From Table 3.4.2, the

friction loss between Section A-A and Section C-C is 0.068m.

(2) Head Loss along Pressure Portion

The computation is carried out from downstream section to upstream section.

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A. Sudden Expansion Loss at Section C-C

The sudden expansion head loss is calculated by

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$$H_{se} = K_{se} \cdot V_1^2 / (2g) = K_{se} \cdot (Q/A_1)^2 / (2g)$$
 $K_{se} = (1 - A_1/A_2)^2$

Where

H_{se}: sudden expansion loss (m)

K_{se}: sudden expansion loss coefficient

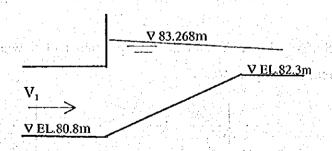
V₁: velocity before sudden expansion (m/s)

Q: discharge (m³/s)

A₁: sectional flow area before sudden expansion (m²)

A₂: sectional flow area after sudden expansion (m²)

g: gravity acceleration



The cross section parameters and the velocity before expansion are

$$A_1 = 2.0 \times 2.0 = 4.0 \text{ m}^2$$

 $A_2 = 2.0 \times 2.468 = 4.936 \text{ m}^2$

Therefore, the sudden expansion head loss at Section C-C becomes as follows.

$$K_{se} = (1 - 4.0/4.936)^2 = 0.036$$

 $H_{se} = 0.036 \times (3/4)^2 / 19.6 = 0.001 \text{(m)}$

B. Friction Head Loss between Section C-C and Section F-F

The friction loss along the pressure portion is the sum of the loss between Section C-C and Section D-D, the one between Section D-D and Section E-E and the one between Section E-E and Section F-F.

The following formula is used for the calculation of the friction loss:

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$$H_f = K_f \cdot L/R \cdot V^2/(2g) = K_f \cdot L/R \cdot (Q/A)^2/(2g)$$

 $K_f = 2g \cdot n^2/R^{1/3}$

Where

H_f: friction head loss (m)

K_f: friction coefficient

n : roughness coefficient

L : length of reach(m)

V : flow velocity (m/s)

Q : discharge (m³/s)

A: flow area (m²)

S : wetted perimeter (m)

Regard: hydraulic radius (m)

g : gravity acceleration

The parameters concerned are given as follows:

Section D-D for the portion between Section C-C and Section D-D:

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$$L = 1.925 + 3.206 = 5.131 \text{ m}$$

$$A = 2.0 \times 2.0 = 4.0 \text{ (m}^2\text{)}$$

$$S = 2 \times 2.0 + 2 \times 2 = 8.0 \text{ (m)}$$

$$R = A/S = 4.0/8.0 = 0.5$$
 (m)

$$V = Q/A = 3.0/4.0 = 0.75$$
 (m/s)

Therefore, the friction head loss between Section C-C and Section D-D is

$$K_{fi} = 19.6 \times 0.0125^2 / 0.5^{1/3} = 0.004$$

 $H_{fi} = 0.004 \times 5.131 / 0.5 \times 0.75^2 / 19.6 = 0.001 \text{ m}$

Section E-E for the portion between Section E-E and Section F-F:

where
$$M$$
 and $L=2.075~m$ and the argument belongs the region of the second M

$$A = 3.1 \times 5.1 = 15.81 \text{ (m}^2)$$

$$S = 2 \times 3.1 + 2 \times 5.1 = 16.4$$
 (m)

$$R = A/S = 15.81/16.4 = 0.964 (m)$$

$$V = Q/A = 3.0/15.81 = 0.190$$
 (m/s)

Therefore, the friction head loss between Section E-E and Section F-F is

$$K_{12} = 19.6 \times 0.0125^2 / 0.964^{1/3} = 0.003$$

$$H_{12} = 0.003 \times 2.075 / 0.964 \times 0.19^2 / 19.6 = 0.00001 = 0.000 \text{ m}$$

The formula for gradual contraction is used for calculating the friction loss of the portion between Section D-D and Section E-E, that is

$$H_{I3} = 2.37n^2LQ^2(1/D_2^{-13/3}-1/D_1^{-13/3})/(D_1-D_2)$$

Where

H_B: friction head loss (m)

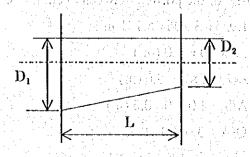
n: roughness coefficient

D₁: equivalent diameter before contraction (m)

D₂: equivalent diameter after contraction (m)

L: length (m)

Q: discharge (m³/s)



The length of the portion between Section D-D and Section E-E, L = 1.55 m, and equivalent diameters are

$$D_1 = (4A/\pi)^{1/2} = (4 \times 15.81/3.1416)^{1/2} = 4.487 \text{ m}$$

 $D_2 = (4A/\pi)^{1/2} = (4 \times 4.0/3.1416)^{1/2} = 2.257 \text{ m}$

Therefore, the friction head loss between Section D-D and Section E-E becomes as follows.

$$H_{\rm p}$$
=2.37×0.0125²×1.55×3.0²×(1/4.487^{13/3}-1/2.257^{13/3})/(4.487-2.257)
=0.00004 m

Hence, the friction head loss along the whole pressure portion is $H_{fro} = 0.001 + 0.00001 + 0.00004 = 0.001m$

C. Bending Loss between Section C-C and Section D-D

The miter bending loss between Section C-C and Section D-D is computed by converting the rectangular cross section into a circular section. The following formula is employed for the bending loss.

$$H_b = K_b V^2/(2g)$$

 $K_b = 0.946 \sin^2(\alpha/2) + 2.05 \sin^4(\alpha/2)$

Where

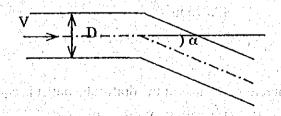
H_b: bending head loss (m)

K_b: bending loss coefficient

V : flow velocity (m/s)

α: bending angle (°)

g: gravity acceleration



Since the section area is A=4.0m², then the equivalent diameter of a

circular section is determined as follows:

$$D=(4A/\pi)^{1/2}=(4\times4.0/3.1416)^{1/2}=2.257 \text{ m}$$

When the bending angle $\alpha=30^{\circ}$ and the coefficient is applied $K_b = 0.946 \times \sin^2(30/2) + 2.05 \times \sin^4(30/2) = 0.073$

Therefore, the bending loss between Section C-C and Section D-D is $H_b = 0.073 \times (3/4)^2 / 19.6 = 0.002 \text{ m}$

D. Loss at Draft-tube Outlet

The loss at the draft-tube outlet is calculated by the following formula for sudden expansion loss:

$$H_{se} = K_{se}V_1^2/(2g)$$

 $K_{se} = \{1-A_1/A_2\}^2$

Where

H_{se}: loss due to sudden expansion (m)

K_{se}: sudden expansion loss coefficient

A₁: flow area at draft-tube outlet (m²)

A₂: flow area after sudden expansion (m²)

V₁: velocity at draft-tube outlet (m/s)

g: gravity acceleration

The area and velocity at the draft-tube outlet and the area after sudden expansion are as follows

$$A_1 = \pi D^2/4 = 3.1416 \times 1.4^2 = 1.539 \text{ m}^2$$

$$A_2 = 5.1 \times 4.25 + 0.5 \times (2.0 + 15.1) \times 5.0 - 2.0 \times 0.5 \times 1.0 \times 1.925 + 1.0 \times 2.0$$

$$= 42.275 \text{ m}^2$$

$$V_1 = Q/A_1 = 3.0/1.539 = 1.949 \text{ m/s}$$

Therefore, the loss at the draft-tube outlet is equal to

$$K_{sc} = (1-1.539/42.275)^2 = 0.964 \text{ (m)}$$

$$H_{lv} = 0.964 \times (3/1.539)^2 / 19.6 = 0.187 \text{ (m)}$$

E. Gradual Contraction Loss between Section D-D and Section E-E

$$H_{gc} = f_{gc} \cdot V_2^2 / (2g) = K_{gc} \cdot (Q/A_2)^2 / (2g)$$

Where

H_{sc}: gradual contraction loss (m)

 f_{gc} : gradual contraction loss coefficient, it is a function of the contraction angle θ and the ratio of the cross sectional areas of the flow before and after contraction, A_2/A_1

V₂: velocity after contraction (m/s)

Q : discharge (m³/s)

A₁: sectional flow area before contraction (m²)

A₂: sectional flow area after contraction (m²)

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The contraction angle, θ , is

$$\theta$$
 = 2arctan[(5.1-2)/2/5]=34.447°

The areas of Section E-E and Section D-D are $A_1=15.81 \text{ m}^2$ and $A_2=4.0 \text{ m}^2$, thus

$$A_2/A_1 = 4.0/15.81 = 0.253$$
m

From the above-mentioned figure,

$$f_{gc} = 0.05$$
, when $A_2/A_1=0.2$; and

$$f_{gc} = 0.04$$
, when $A_2/A_1 = 0.3$

Thus the factor for $A_2/A_1=0.253$ can be obtained by

$$f_{sc} = (0.04-0.05) \times (0.253-0.2) / (0.3-0.2) + 0.05 = 0.045$$

Therefore, the gradual contraction loss between Section E-E and Section

D-D is calculated as follows

$$H_{gc} = 0.045 \times (3/4)^2 / 19.6 = 0.001 \text{ m}$$

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(3) Summary of Head Loss of Tailrace Portion

Assuming allowance of 4.0% for the sum of the loss of the non-pressure

portion and loss of the pressure portion, the head loss of the whole tailrace is summarized as follows

Pressure Portion:

Draft-tube outlet: 0.187 m

Friction: 0.001 m

Bending: 0.002 m

Gradual contraction: 0.001 m

Sudden expansion: 0.001 m

Non-pressure Portion:

Friction: 0.068 m

Exit: 0.307 m

Allowance (4%): 0.020 m

Total: 0.587 m

6) Summary of Head Loss

The whole head losses from the intake to the tailrace in case of Q=3m³/s are summarized in Table 3.4.3. It can be found that the total head loss along the outlet facilities is 1.608m.

3.4.3 Effective Head

The effective head can be obtained by

 $H = H - \Sigma H$

Where

H, : effective head (m)

H_g : gross head (m), H_g =NWL-TWL

 ΣH_L : total head loss

From the aforementioned sections, NWL, TWL and total head loss of the generation system for Q=3 m³/s are

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NWL: 148.9 m

TWL: 82.91 m

 ΣH_L : 1.69m

Therefore, the effective head corresponding to Q=3.0 m³/s is

 $H_e = NWL-TWL - \sum H_L = 148.9-82.91 - 1.69 = 64.30 \text{ m}$

Table 3.4.1 Stage Discharge Table of River Reach Downstream of Spillway

Remarks			3 m ³ /s	6 m ³ /s					31 m ³ /s					85 m ³ /s		· 2		100 yr Flood					PMF outflow	PMF inflow	
Discharge Q (m3/s)	0.0	1.9	3.0	6.0	6.1	11.9	19.2	27.8	31.0	37.6	48.5	60.4	73.4	85.0	87.2	169.6	271.2	340.0	389.9	523.9	834.1	1195.3	1310.0	1600.0	1603.8
Manning coef. n	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03	0.03
1,12	0.112	0.112	-0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112	0.112
Slope	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125	0.0125
R ^{2/3}	0.00	0.21	0.26	0.34	0.34	0.44	0.53	0.62	0.64	69.0	0.76	0.83	0.89	0.95	96.0	1.23	1.46	1.58	1.66	1.84	2.15	2.42	2.49	2.65	2.66
Hydraulic radius R (m)	00.0	0.10	0.13	0.20	0.20	0.29	0.39	0.48	0.51	0.58	0.67	0.76	0.85	0.92	0.93	1.36	1.76	1.98	2.13	2,49	3.16	3.77	3.94	4.32	4.33
Wetted perimeter S (m)	24.0	24.2	24.3	24.4	24.4	24.7	24.9	25.1	25.2	25.3	25.6	25.8	26.0	26.2	26.2	27.4	28.5	29.1	29.6	30.7	32.9	35.2	35.8	37.4	37.4
Area A (m2)	0.0	2.4	3.1	4.8	4.8	7.2	2.6	12.1	13.0	14.6	17.0	19.5	22.0	24.1	24.5	37.1	50.0	57.8	63.1	76.5	104.0	132.5	141.0	161.7	162.0
Surface width B2 (m)	24.0	24.1	24.1	24.2	24.2	24.3	24.4	24.5	24.5	24.6	24.7	24.8	24.9	25.0	25.0	25.5	26.0	26.3	26.5	27.0	28.0	29.0	29.3	30.0	30.0
Bottom width B1 (m)	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0	24.0
Depth h (m)	0.000	0.100	0.131	0.198	0.200	0.300	0.400	0.500	0.534	0.600	0.700	0.800	0.900	0.984	1.000	1.500	2.000	2.298	2.500	3.000	4.000	5.000	5.292	5.991	6.000
Elevation (m)	000.0	0.100	0.131	0.198	0.200	0.300	0.400	0.500	0.534	0.600	0.700	0.800	0.900	0.984	1.000	1.500	2.000	2.298	2.500	3.000	4.000	2.000	5.292	5.991	0.000

Table 3.4.2(1/2) Non-uniform Computation of Tailrace by Direct Step Method
Discharge

				EL 82.631			1.	Remarks	Sec. A-A										y'							Sec. B-B				
			∢	٥			Surface	elevation (m)	82.912	82.922	82.932	82.952	82.962	82.972	82.982	82.932	83,012	83.022	83.032	83,042	83.052	83.072	83.082	83.092	83.102	83,110	83.143	33.168	83.187	83.202
	-			Tailrace				(m)	0000	0.072	0.291	1 189	1.875	2.724	3.740	4.928 6.292	7.835	9.562	11.476	13.582	0 00 00 0 00 00 0 00 00 0 00 00	21.095	24.011	27.140	30.486	33.036	33,239	33.473	33.729	34.000
		:		 	3			S S		0.072	0.219	0.526	0.686	0.849	1.017	363	1.543	1.727	1914	2.106	2,302	2.707	2.916	3,129	3,346	2.550	0.203	0.234	0.256	0.282
				90 12	0 1 1 1			Ĭ <u>-</u> Š		-0.0033	-0.0032	0000	-0.0028	-0.0027	-0.0026	0.0020	-0.0023	-0.0022	-0.0021	0.0020	02000	-0.0018	-0.0018	-0.0017	-0.0016	-0.0018	-0.2792	-0.2788	0.2785	-0.2783
		Air Shaft	œ		\ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \ \		Average slope	of energy grade line Sr		0.0033	0.0032	0.0000	0.0028	0:0027	0.0026	0.0025	0,0023	0,0022	0.0021	0.0020	0.0020	0.0018	0.0018	0.0017	0.0016	0.0016	0.0014	0.0010	0.0008	0.0007
] - -		o		tine S=n²V	0.0034	0.0033	0.0031	0.0030	0.0027	0.0026	0.0025	0.0024	0.0022	0.0021	0.0021	0.0020	00019	0.0018	0.0017	0.0017	0.0016	0.0016	0.0012	60000	0,0007	0.0000
			ရာ ပု		5 EL 80.8			ΔĒ		-0.0002	000 000 000 000 000 000 000 000 000 00	- 4	-0.0019	-0.0023	-0.0026	0.0029	-0.0035	-0.0038	-0.0040	-0.0043	5000	-0.0049	-0.0051	-0.0053	-0.0055	-0.0040	-0.0567	-0.0654	-0.0713	-0.0784
			Between Section A-A and Section B-B Between Section B-B and Section C-C	(Final)	(Final)			Specific onergy E(m)	0.918	0.919	0.919	0.92	0.924	. 0.926	0.929	0.932	0.00	0.942	0.946	0.951	0.955	0.965	0.970	0.975	0.981	0.985	1,041	1,107	1.78	1.332
			section A-A	0.007	0.038		Velocity	head α $\sqrt{^2}/2g(m)$	0.306	0.296	0.287	0.270	0.262	0.254	0.247	0.239	0.226	0.220	0.214	0.208	0.203	0.192	0.188	0.183	0.178	0.175	0.142	0.117	0.099	0.084
E			Between S Between S	(Initial)	(Initial)	EE		Velocity V(m/s)	2,450	2.410	2.372	2,000	2.265	2.231	2.198	2.167	2.106	2.077	2.048	2.021	1.994	1942	1.917	1,893	1.870	1.853	1.668	1.516	1.389	1,283
2.0	0.1	0.0125	0.0	0.01	0.00	0.612	Hydraulic	radius R(m)	0.380	0.384	0.387	2000	0.398	0:402	0.406	0.409	0,416	0.419	0.423	0.426	0.429	0.436	0.439	0.442	0.445	0.447	0.474	0.497	0.519	0.557
#	ll Ø	li C	П П	∆ h=	_4∆	표 ^함	Wetted	perimette r P(m)	3.225	3.245	3.265	2020	3.325	3,345	3.365	3,385	3,425	3,445	3,465	3,485	3.505	3.545	3.565	3.585	3.605	3,619	3,799	3.979	4.159	4,339
ctangular on	rgy factor	coefficier	9 6	n depth r I=0 (m)	n depth	_ \$		Area A(m²)	1,225	1.245	1.265	1 205	1,325	1,345	1.365	1.385	1.425	1.445	1.465	1.485	1,505	1545	1.565	1.585	1.605	1.619	1,799	1.979	2.159	2.339
Width of rectangular cross section	Kinetic energy correction factor	Roughness coefficier	Bottom slope Bottom slope	Increment in depth direction for I=0 (m	Increment in depth direction for ICO (m	Initial depth Critical depth		Depth h(m)	0.612	0.622	0.632	0.042	0.662	0.672	0.682	0.692	0.712	0.722	0.732	0.742	0.752	0.772	0.782			0.810	0.900	0800	80: 80:	1.170
		-	_				Bottom	elevation (m)	82.3	82.3	82.3	0 0 0 0 0 0 0 0	82.3	82.3	82.3	823	82.3	82.3	82.3	82.3	82.0	2 6	82.3	82.3	82.3	82.3	82.24	82.18	82.11	82.03
					ŧĄ)			2 2	o		۲۷ ۲	ວ ຊ	· v	9	~		, <u>c</u>	÷	12		4 0	. 4	7	82	5	50	7	. 22	23	24 25

Table 3.4.2(2/2) Non-uniform Computation of Tailrace by Direct Step Method

	▽ EL 82.631	
	<u></u>	₹
	Tailrace	
	7 EL 82.3	
	Y	a
Air Shaft		
	0	308
Sion B-B	ction C-C	inal) CEL
oction A-A and Section B-B	B-B and Section ((i.)
(s) () (see Section	<u> </u>	al) 0.03 }
3.0 (m², 2.0 (m 1.0 (m 125 0.0 Betwe	78 Betwe 3.01 (Initi),09 (Initial) 12 (m) 12 (m)
	-0.27	90
gular 8= 8 = 6 = 6 = 6 = 6 = 6 = 6 = 6 = 6 =	3) (E) (E)	(m) (m) H= ho=
Discharge $\Omega=3.0 \ (m^3/s)$ Width of rectangular $B=2.0 \ (m)$ cross section Kinetic energy $\alpha=1.0$ correction factor Roughness coefficier $n=0.0125$ Bottom slope $\Omega=0.00125$	Bottom slope Increment in dep direction for I=0	Increment in depth $\Delta h=$ 0.09 (Initial) direction for I<0 (m) H= 0.612 (m) Critical depth ho= 0.612 (m)

	-		-				-							_
Remarks			1					L **			***			-
Surface elevation (m)	83.223	83.231	83.237	83.242	83.247	83,251	83.254	83.257	83.260	83,262	83.264	83.266	83.267	02000
(m) (m)	34.572	34.868	35.169	35,473	35,781	36.091	36,403	36.716	37.031	37,347	37,664	37,982	38,300	367.96
W) W)	0.290	0,296	0.301	0.305	0.308	0.310	0.312	0.314	0.315	0.316	0.317	0.318	0.318	125
<u>'S</u> -1	-0.2782	-0.2782	-0.2781	-0.2781	-0.2780	-0.2780	0.2780	-0.2779	-0.2779	-0.2779	-0.2779	-0.2779	-0.2779	-02770
Average slope of energy grade line Sr	0.0004	0.0004	0.0003	0.0003	0.0002	0.0002	0.0002	0.0002	0.0001	0,0001	0.0001	0.0001	0,0001	10000
Slope of energy grade line S,=n²√ ²/R⁴/³	0.0004	0.0003	0.0003	0.0003	0.0002	0.0002	0,0002	0.0002	0.0001	0.0001	0.0001	0.0001	0.0001	0.0001
ΔĒ	-0.0807	-0.0824	-0.0837	-0.0847	-0.0855	-0.0862	-0.0867	-0.0871	-0.0875	-0.0878	-0.0881	-0.0883	-0.0885	-0.0376
Specific energy E(m)	1,413	1.495	1,579	1.663	1.749	1.835	1.922	2.009	2.096	2.184	2.272	2,361	2.449	2.487
Velocity head α $V^2/2g(m)$	0.063	0.055	0.049	0.044	0.039	0.035	0.032	0.029	0.027	0.025	0.023	0.021	0.019	0.019
		.042	0.981	0.926	0.877	0.834	0.794	0.758	0.725	0,695	0.667	0.641	0.617	0.608
Hydraulic radius R(m)	0.574	0.590	0.605	0.618	0.631	0.643	0.654	0.664	0.674	0.683	0.692	0.701	0.708	0,712
ከተ	4.699	4.879	5.059	5.239	5.419	5.599	5.779	5.959	6.139	6.319	6,499	6.679	6.859	6.935
Area A(m²)	2.699	2.879	3,059	3,239	3.419	3.599	3,779	3.959	4.139	4.319	4.499	4.679	4.359	4.935
Depth h(m)	1.350	7.440	056.	1.620	1.710	1.800	1.890	1.580	2.070	2.160	2.250	2.340	2.430	2.468
Bottom elevation (m)	20.00	87.78	× × ×	81.62	40.10	81.45	31.36	81,28	81.19	81.10	5.0	80.93	80.84	80.80
Š	\$ 5	77	3 8	3 8	3 2	e d	32	3	4 1	3 :	e (æ (33
		-	_	_	_	-			_	_	_		_	_

Notes: E=h+ \alpha \subseteq^2/2g(m)

Average slope of energy grade line Sr= 0.5(n²V₁²/R₁^{4/3}+n²V₂²/R₂^{4/3})

Flow depth at starting section is set to equal the critical depth, hc=(α Q²/gB²)^{1/3} Tailwater level = 82.631m

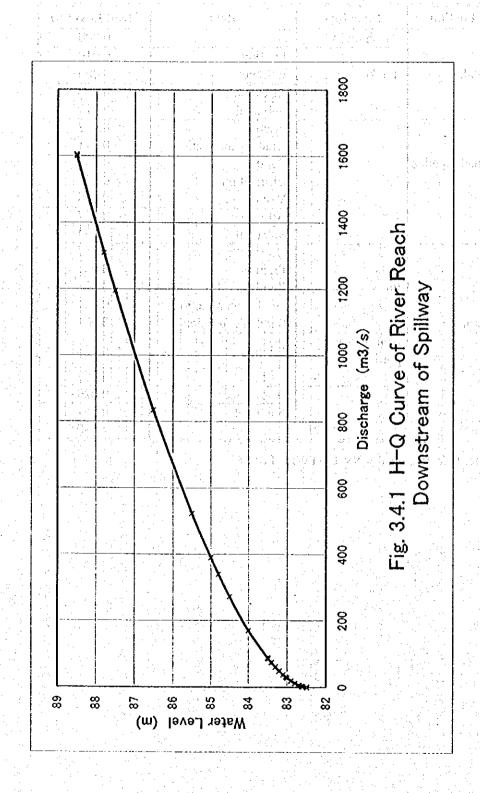
Ninimum specific energy Ec=1.5ha≕ 0.918 m <E₆<E₆ Subcritical flow Head loss H_U≂ E₁-E₂+z₁-z₂

0.068 Between Section A-A and Section C-C 0.066 Between Section A-A and Section B-B 0.002 Between Section B-B and Section C-C

Table 3.4.3 Summary of Head Loss (Q=3 m³/s and NWL=EL.148.9m)

Portion	Structure	Item	Head Loss (m)			
	Trash racks		0.000			
		Friction	0.010			
Intake	Intake	Entrance	0.006			
		Allowance	0.002			
		Sub total	0.019			
		Friction	0.918			
		Gradual contraction	0.011			
Steel pipeline		Bend	0.082			
		Bifurcation	0.006			
		Valve	0.000			
		Allowance	0.067			
		Sub total	1,084			
		Draft-tube	0.187			
		Friction	0.001			
	Pressure portion	Bend	0.002			
		Gradual contraction	0.001			
Tailrace		Sudden expansion	0.001			
	Non-pressure portion	Friction	0.068			
		Exit	0.307			
		Allowance	0.020			
		Sub total	0.587			
	Total		1.690			

Note: Effective head = NWL - TWL - Loss = 64.300



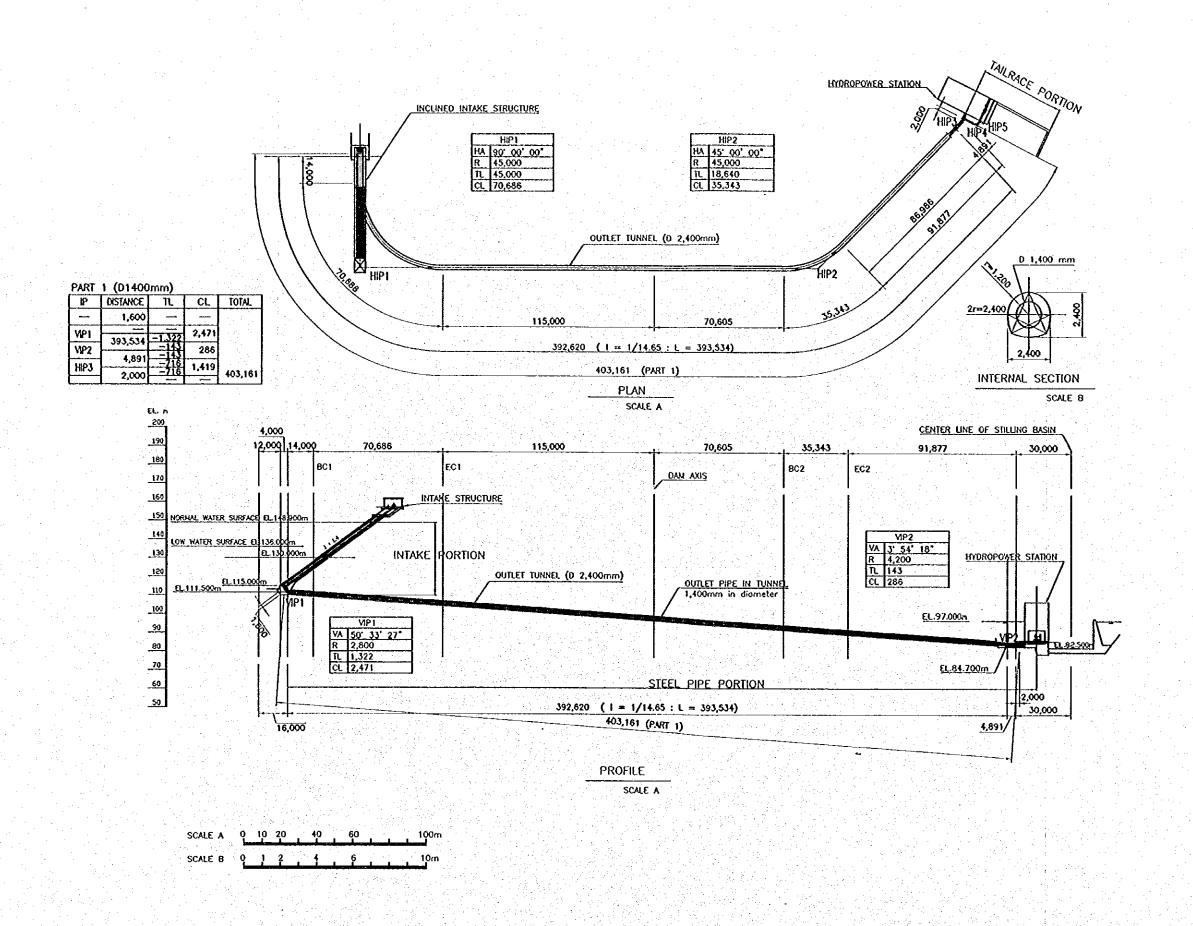
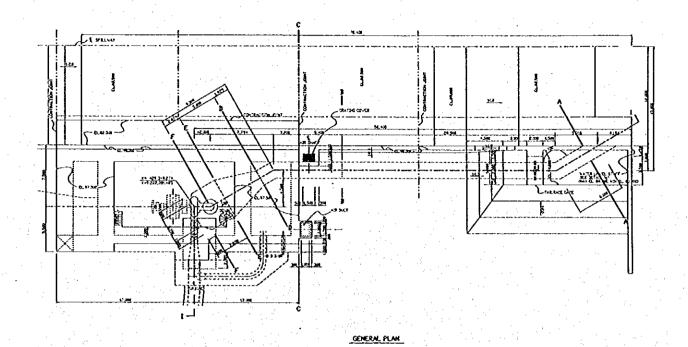


Fig. 3.4.2 CALCULATION PORTIONS FOR HEAD LOSS



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Flg. 3,4,3 Typical Sections For Loss Calculation