

REPUBLIC OF INDONESIA
MINISTRY OF COMMUNICATIONS
DIRECTORATE GENERAL OF LAND TRANSPORT
AND INLAND WATERWAYS

TENDER DOCUMENTS
FOR
NEW RAILWAY LINE FOR CENGKARENG AIRPORT
CONSTRUCTION PROJECT

STRUCTURAL CALCULATION SHEETS

PACKAGE I CIVIL AND ARCHITECTURAL WORK

10 of 11

AUGUST 1984

JAPAN INTERNATIONAL COOPERATION AGENCY
(JICA)



RY

国際協力事業団	
受入 月日 84.11.19	108
登録No. 10882	616
	SDF

マイクロ
フロッピー作成

STRUCTURAL CALCULATION SHEETS
CONTENTS

- 1 OF 11
§§1. P. C. GIRDERS
§§2. R. C. GIRDERS
- 2 OF 11
§§3. PIERS
§§4. ABUTMENTS
- 3 OF 11
§§5. VIADUCT V047
§§6. VIADUCT V048
- 4 OF 11
§§7. VIADUCT V089
- 5 OF 11
§§8. VIADUCT V094
- 6 OF 11
§§9. VIADUCT OF PLATFORM VP2
§§10. VIADUCT OF PLATFORM VP5
§§11. R. C. GIRDER OF PLATFORM RCP1
- 7 OF 11
§§12. VIADUCT V129
- 8 OF 11
§§13. BOX CULVERTS
- 9 OF 11
§§14. BUILDINGS
- 10 OF 11
§§15. CALCULATION OF MECHANICAL
FOR AIRPORT TERMINAL STATION,
KOTA INTAN STATION AND SIGNAL
CABINS
§§16. LIGHT INTENSITIES (LUX)
- 11 OF 11
§§17. SUPPORTING STRUCTURE FOR
OVERHEAD CONTACT SYSTEM

JICA LIBRARY



1034284[8]

§§ 15. CALCULATION OF MECHANICAL
FOR
AIRPORT TERMINAL STATION,
KOTA INTAN STATION AND
SIGNAL CABINS

PAGE

M-1 ~ M-6	General Conditions	(VAC)
M-7 ~ M-19	Calculation Sheets Terminal BLDG	(")
M-20 ~ M-26	Ditto Booking Office	(")
M-27 ~ M-31	Ditto Signal Cabin	(")
M-32 ~ M-42	Ditto Kata Intan Station	(")
M-43 ~ M-51	Calculation Sheets Airport Station Kata Intan Station	(Plumbing)

AIR CONDITIONING AND VENTILATION

A. DESIGN CONDITIONS

AIR CONDITIONING :

	DB °C	WB °C	RH %	X kg/kg
Outdoor	32.5	27.0	65	0.0203
Indoor	26	18.8	56	0.0105
	Δt 6.5			ΔX 0.0093

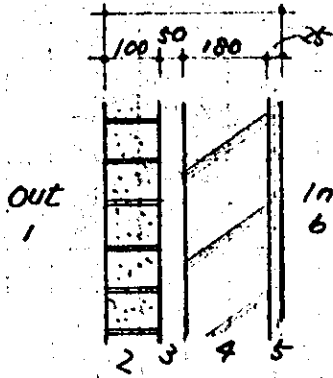
VENTILATION :

	Air Changes (times/Hr)
Lavatory (for Staff)	15
Lavatory (for passenger)	20
Mechanical Room	10
Power Room	15
Kitchen	30
Ablution Room	10
Musholla	15
Maintenance & Janitor's Room	10
Porter's Room	10
Signal Telcom Device Room	10
Storage	10

B. AIR CONDITIONING COOLING LOAD

1. Coefficients of Transmission (K) of Walls & Roofs

"K" of Walls



Construction	t	λ	R
1. Outside Air	-	-	0.025
2. Brick	0.1	0.53	0.188
3. Air space	-	-	0.18
4. Concrete	0.18	1.4	0.128
5. Plaster	0.025	0.53	0.047
6. Inside Air	-	-	0.125

Total R = 0.693

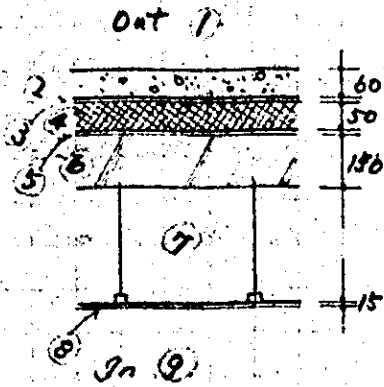
$K = 1/0.693 \approx 1.45 \rightarrow$ Wall Group C*

p=26.9

* Refrn, Table 6 of ASHRAE FUNDAMENTALS (81)

"K" of Roofs

Roof Type - 1

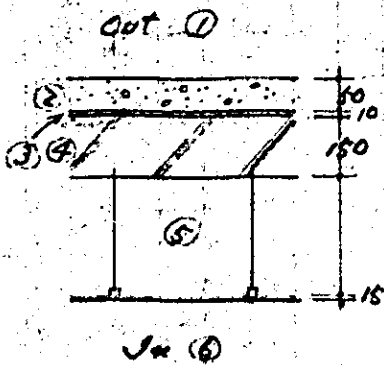


Construction	t	λ	R
1. Outside Air	-	-	0.025
2. Mortar	0.06	1.2	0.05
3. Asphalt & Roofing	0.015	0.12	0.125
4. Insulation	0.05	0.035	1.428
6. Concrete	0.15	1.4	0.107
7. Air space	-	-	0.2
8. Ceiling Board	0.015	0.06	0.25
9. Inside Air	-	-	0.125

Total R = 2.31

$K = 1/2.31 = 0.44$ Roof NO. 12 (ASHRAE)

Roof Type - 2



Construction	t	λ	R
1. Outside Air	-	-	0.025
2. Mortar	0.05	1.2	0.041
3. Asphalt & Roofing	0.01	0.12	0.083
4. Concrete	0.15	1.4	0.107
5. Air space	-	-	0.2
6. Inside Air	-	-	0.125

$K = 1/0.581 = 0.86$ Roof NO. 3 (ASHRAE) Total R = 0.581

"K"-value of partition Wall, Floor & Others

partition Wall : $K = 2.5$ (Concrete brick, plaster)

Floor : $K = 2.0$ (Concrete, Terrazzo)

Door (outside) : $K = 2.7$ (Steel-klash type)

Glass of window : $K = 5.5$ (single 6mm)

2. Rates Heat Gain from Occupants of Conditioned Spaces

Typical Application	Degree of Activity	SH (Kcal/Hr)	LM (Kcal/Hr)
Offices (1)	Seated, Very Light Work, Writing	58	98
" (2)	Seated, Light Work, Typing	65	65
Others	Light Bench-work	87	110

3. Cooling Load Temperature Differences (CLTD)

CLTD corr of Roofs {ASHRAE HANDBOOK (1981), CHAPTER 26}

$$\{ (CLTD + LM) \times K + (25.5 - TR) + (T_o - 29.9) \} \times f$$

Description	15:00	17:00	19:00
Roof Type-1 CLTD (Table 5) LM (" 9) $K=1, 25.5 - TR = -0.5$ $T_o - 29.9 = 3.1, f=1$	16 -0.5 -0.5 3.1 18.1 $19.0^{\circ}C$	18 -0.5 -0.5 3.1 20.1 $21.0^{\circ}C$	19 -0.5 -0.5 3.1 21.1 $22.0^{\circ}C$
Roof Type-2 CLTD (Table 5) LM (" 9) $K=1, 25.5 - TR = -0.5$ $T_o - 29.9 = 3.1, f=0.5$	31 -0.5 -0.5 3.1 33.1 $17.5^{\circ}C$	36 -0.5 -0.5 3.1 38.1 $19.5^{\circ}C$	39 -0.5 -0.5 3.1 36.1 $18.5^{\circ}C$

CLTD_{corr} of Wall {ASHRAE HANDBOOK (1981), CHAPTER 26}

$$CLTD_{corr} = (CLTD + LM)K + (25.5 - TR) + (T_o - 29.4)$$

where CLTD ~ Table 7, LM = Table 9, K = 1

TR = 26°C, T_o = 32.5°C ; Wall Group No. C

Time Orientation	15:00	17:00	19:00
N	7 °C	8 °C	10 °C
E	18 "	19 "	18 "
S	17 "	20 "	22 "
W	10 "	12 "	14 "

CLTD_{corr} & Solar Heat Gain of Glass

$$CLTD_{corr} = CLTD + (25.5 - TR) + (T_o - 29.4) \dots \text{from Table 10}$$

15:00 ~ 106°C, 17:00 ~ 96°C, 19:00 ~ 86°C

Solar Heat Gain (SC x SHGT x CLF)

where SC = 0.78, SHGT ~ Table 11, CLF ~ Table 13 & 14

Time Orientation	15:00	17:00	19:00
N	64 Kcal/m ²	63 Kcal/m ²	51 Kcal/m ²
E	167 "	133 "	98 "
S	159 "	119 "	85 "
W	230 "	222 "	236 "

AIRPORT TERMINAL STATION

M-7

STATION BLDG. No. Temp. (s)

No. (V) 4

L 63

Area

25

25

25

Volume

63

3

No. 1

WAITING

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
W=N	Outside W.	1.25	6.1	7	8	10	62	71	89			
S=N.	"	1.25	15.8	7	8	10	161	184	230			
W=N	Window	5.5	3.9	10.6	9.6	8.6	228	206	185			
	Parti- -tion W	2.5	10	6.5	3.25	3.25	82	52	32			
	Window											
	Floor											
	Roof											
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		
	Area (m ²)	Factor	m ²								c/h	
W=N			3.9	64	63	51	250	246	199		m ³ /h	
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss G. Total		
	Incandescent lamp			KV								
	Fluorescent lamp	20	20/m ²	0.5' KV		1000	510	510	510			
	Meter 0.08KV - 0.37KV			KV								
	0.37KV - 1.5KV			KV								
										Supply Air Temp. Difference ° deg		
										Latent Heat		
	Gas equip. the other											
Internal Heat (Equipment) Sub. Total												
	Person	25' x 0.2 = 8 (person)		8 x 50			464	464	464	8 x 48	384	
	Infiltration	6.2 x 0.25 = 1.55	32 m ³ /h		x 6.5 deg x 0.28		59	59	59	715 x Δx	225	
Heat gain Sub. Total							1,816	1,822	1,819	0.0078	609	
Heat gain Grand Total							2,626	2,681	2,428	O.A. 25	m ³ /h Person	
Possible Heat Rate SHR								0.75		200	m ³ /h	
Required Air Volume = $\frac{SH}{0.28 \Delta t}$										m ³ /h		
										Air change		

AIRPORT TERMINAL STATION

M-8

Story STATION BLDG. Rm. Temp. (°C) °C (°F) R.H. (%) % No. 2
 Rm. Name STATION MASTER Rm. (W) 5 x L 3.7 = Area 18.5 m² H 2.5 m = Volume 46.3 m³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat Loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
W=N	Outside W.	1.25	10.8	7	8	10	110	126	157			
W=N	Window	5.5	1.7	10.6	9.6	8.6	100	90	81			
	Parti- -tion W	7.5	1.0	3.25	3.25	3.25	82	82	82			
	Window											
	Floor											
	Roof											
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor	m ²									m ³ /h
W=N			1.68	64	63	51	108	106	86			m ³ /h x deg x
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Factor		
Internal Heat (Equipment) Sub. Total										Heat Loss C. Total		
	Incandescent lamp											
	Fluorescent lamp	20	W/m ²	0.37			370	370	370			
	Meter 0.09KW - 0.37KW											
	0.37KW - 1.5KW											
										Supply Air Temp. Difference ° deg		
										Latent Heat		
	Gas equip. the other											
Internal Heat (Equipment) Sub. Total												
	Person			2 x 58			116	116	116	2 x 48		96
	Infiltration	46.3 x 0.5 = 23.15 m ³ /h		6.5 deg x 0.28			42	42	42	7.5 x Δt		162
Heat gain Sub. Total												
Heat gain Grand Total												
Sensible Heat Rate SHR												
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$												
										Air change		

AIRPORT TERMINAL STATION

M-9

STORY STATION BLDG. No. Temp. (s) °C (w) °C R.M. (s) (w) No. 1

Rm. Name OFFICE-1 Rm. (V) 5 n x L 9.3 n = Area 46.5 m², H 2.5 m = Volume 116.3 m³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δ t (deg)			S.H. kcal/h			Δ t deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
E-N	Outside W.	1.00	2.5	7	8	10	97	111	128			
E-N	Window	3.5	3.0	10.6	9.6	8.6	175	159	142			
	Parti- -tion W	2.5	18.8	3.25	3.25	3.25	153	153	153			
	Window											
	Floor											
	Roof											
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		
	Area (m ²)	Factor	m ²								c/h	
E-N			3.0	64	62	51	192	189	153		m ³ /h x deg x	
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss		
	Incandescent lamp			KV							C. Total	
	Fluorescent lamp	20	m ²	0.93	KV		230	230	230			
	Meter 0.09KV - 0.37KV			KV								
	0.37KV - 1.5KV			KV								
										Supply Air Temp. Difference = deg		
										Latent Heat		
	Gas equip. the other			0.5 KW x 860	Kcal/KV		430	430	430			
Internal Heat (Equipment) Sub. Total												
	Person			0 x 50			262	264	264	0 x 40	304	
	Infiltration	116.3 x 0.8 = 93	m ³ /h	1.65 deg x 0.28			170	170	170	7.5 x 40	652	
Heat gain Sub. Total							(2611)	2606	2580	0.000	(1026)	
Heat gain Grand Total							(3697)	3642	3616	O.A. 25	m ³ /h Person	
Sensible Heat Rate SHR							1.1			200	m ³ /h	
Required Air Volume = $\frac{SH}{0.28 \text{ RAE}}$											m ³ /h	
										Air change		

AIRPORT TERMINAL STATION

M-10

Story No. 4
 No. 4
 No. (W) 6.6 = x L 9.5 = Area 29.7 =² H 2.5 = Volume 27.3 =³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
E=N	Outside c.	1.25	12.5	7	8	10						
S=N	"	1.25	7.2	7	8	10	279	318	398			
W=N	"	1.25	6.5	7	8	10						
E=N	Window	5.5	3.0	10.6	9.6	8.6	740	265	327			
S=N	"	5.5	3.9	10.6	9.6	8.6						
	Parti- tion	2.5	10	3.25	3.25	3.25	82	82	82			
	Window											
	Floor											
	Roof											
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor	m ²							m ³ /h		
E=N			3.0	64	63	51	442	435	352	m ³ /h x deg x		
S=N			3.9	64	63	51						
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss C. Total		
	Incandescent lamp			KV								
	Fluorescent lamp	20	4/m ²	0.6	KV		600	600	600			
	Meter 0.09KV - 0.37KV			KV								
	0.37KV - 1.5KV			KV								
										Supply Air Temp. Difference ° deg		
										Latent Heat		
	Gas equip. the other			0.5 KV x 260 KV/h			430	430	430			
Internal Heat (Equipment) Sub. Total												
	Person			4 x 65			260	260	260	4 x 65	260	
	Infiltration	27.3 x 1 x 7.5 = 204.75 m ³ /h		x 6.5 deg x 0.28			137	137	137	7.5 x Δt	526	
Heat gain Sub. Total												
							2627	2506	0.0098	786		
Heat gain Grand Total							3919	3413	3372	O.A. 25	m ³ /h Person	
							3770	3760	3710	100	m ³ /h	
Sensible Heat Rate SHR							0.77				m ³ /h.m ²	
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$											m ³ /h	
										Air change		

AIRPORT TERMINAL STATION

M-11

Story STATION BLDG. No. Temp. (°C) _____ °C (°F) _____ R.H. (g) _____ % (°F) _____
 No. (26) MEETING Ra. (W) $5^2 = x L 9$ a = Area 486 m² H 2.5 = Volume 1215 m³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δ t (deg)			S.H. kcal/h			Δ t deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
N	Outside E.	1.25	21	7	8	10	351	901	501			
E=N		1.25	12.5	7	8	10						
N	Window	5.5	1.5	10.6	9.6	8.6	92	25	76			
	Parti- -tion X	2.5	12.5	2.25	2.25	2.25	110	110	110			
	Window											
	Floor	2.0	45.6	2.25	2.25	2.25	316	316	316			
	Roof	0.86	22.6	17	19.5	18.5	711	816	774			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor	m ²							m ³ /h		
N			1.6	62	63	51	103	101	82	m ³ /h x deg x		
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss E. Total		
	Incandescent lamp											
	Fluorescent lamp	20	4/100	0.98			980	980	980			
	Meter 0.09KV - 0.37KV											
	0.37KV - 1.5KV											
										Supply Air Temp. Difference = deg		
										Latent Heat		
	Gas equip. the other.											
Internal Heat (Equipment) Sub. Total												
	Person		$23.6 \times 0.7 = 15$		16×58		870	870	870	15×48	720	
	Infiltration		$121.5 \times 0.5 = 61$ m ³ /h		$\times 6.5$ deg x 0.28		112	112	112	$712 \times \Delta t$	728	
Heat gain Sub. Total							3,627	3,791	(3,821)	6,000	(1,148)	
Heat gain Grand Total							4,775	4,937	(4,969)	O.A. 25	m ³ /h Person	
Sensible Heat Rate, BHR									0.77	375	m ³ /h	
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$											m ³ /h	
										Air change		

AIRPORT TERMINAL STATION

M-12

Story STATION 0109 Rm. Temp. (a) °C (w) °C R.H. (a) % (w) % No. 6
 Rm. Name (27) POWER INSPECTION Rm. (W) 5.3 m x L 7.8 m = Area 41.7 m² H 2.5 m = Volume 102.5 m³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
E=N	Outside c.	1.45	9.2	7	8	10	94	107	124			
E=N	Window	5.5	1.6	10.6	9.6	8.6	94	85	76			
	Parti- tion Y	2.5	13.3	3.25	3.25	3.25	109	109	109			
	Window											
	Floor	2.0	21.2	3.25	3.25	3.25	270	270	270			
	Roof	0.96	21.2	17	19.5	18.5	606	695	659			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor	m ²								m ³ /h	
E=N			1.6	69	63	51	103	101	82		m ³ /h x deg x	
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss C. Total		
	Incandescent lamp											
	Fluorescent lamp	20	0.53				230	230	230			
	Meter 0.09KW - 0.37KW											
	0.37KW - 1.5KW											
Latent Heat										Supply Air Temp. Difference ° deg		
	Gas equip.											
	The other		0.5 x 660				430	430	430			
Internal Heat (Equipment) Sub. Total										Latent Heat		
	Person			4 x 50			232	232	232	4 x 40		192
	Infiltration	102.5 x 0.5 = 51.25 m ³ /h					95	95	95	7.5 x Δx		365
Heat gain Sub. Total										Sub. Total		
Heat gain Grand Total										Sub. Total		
Sensible Heat Rate SHR										Sub. Total		
Required Air Volume = $\frac{SH}{0.28 \Delta t}$										Sub. Total		
										Air change		

AIRPORT TERMINAL STATION

M-13

STORY STATION: OLDY Rm. Temp. (s) °C (w) °C R.H. (g) % (w) %
 Rm. Name: SIGNAL TELCOM Rm. (W) 5' x L 70' = Area 350 m², H 2.5 = Volume 875 m³ No. 7

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
E=N	Outside c.	1.45	70	7	0	10	75	85	106			
E=N	Window	5.5	52	10.6	9.6	8.6	308	275	246			
	Partly-shaded	2.5	12.5	3.25	3.25	3.25	102	102	102			
	Window											
	Floor											
	Roof	0.86	39	17	19.5	18.5	571	655	621			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor	m ²									m ³ /h
E=N			16	62	63	51	105	101	82			m ³ /h x deg x
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss G. Total		
	Incandescent lamp			KV								
	Fluorescent lamp	20	4/m ²	0.74 KV			280	280	280			
	Meter 0.09KW - 0.37KW			KV								
	0.37KW - 1.5KW			KV								
Gas equip. the other										Latent Heat		
			1.5 x 260				1290	1290	1290			
Internal Heat (Equipment) Sub. Total												
	Person			4 x 65			260	260	260	4 x 65		260
	Infiltration	39 x 0.5 = 20 m ³ /h		x 6.5 deg x 0.28			37	37	37	7.52 / Δt		191
Heat gain Sub. Total												
Heat gain Grand Total												
Sensible Heat Rate: SHR												
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$												
										Air change		

AIRPORT TERMINAL STATION

M-14

Story STATION BLDG Rm. Temp. (°C) °C (F) °C, R.H. (F) % (W) No. 8
 Rm. Name REST Rm. (W) 6.8 m x L 2.8 m = Area 53.1 m² H 2.5 m = Volume 132.8 m³

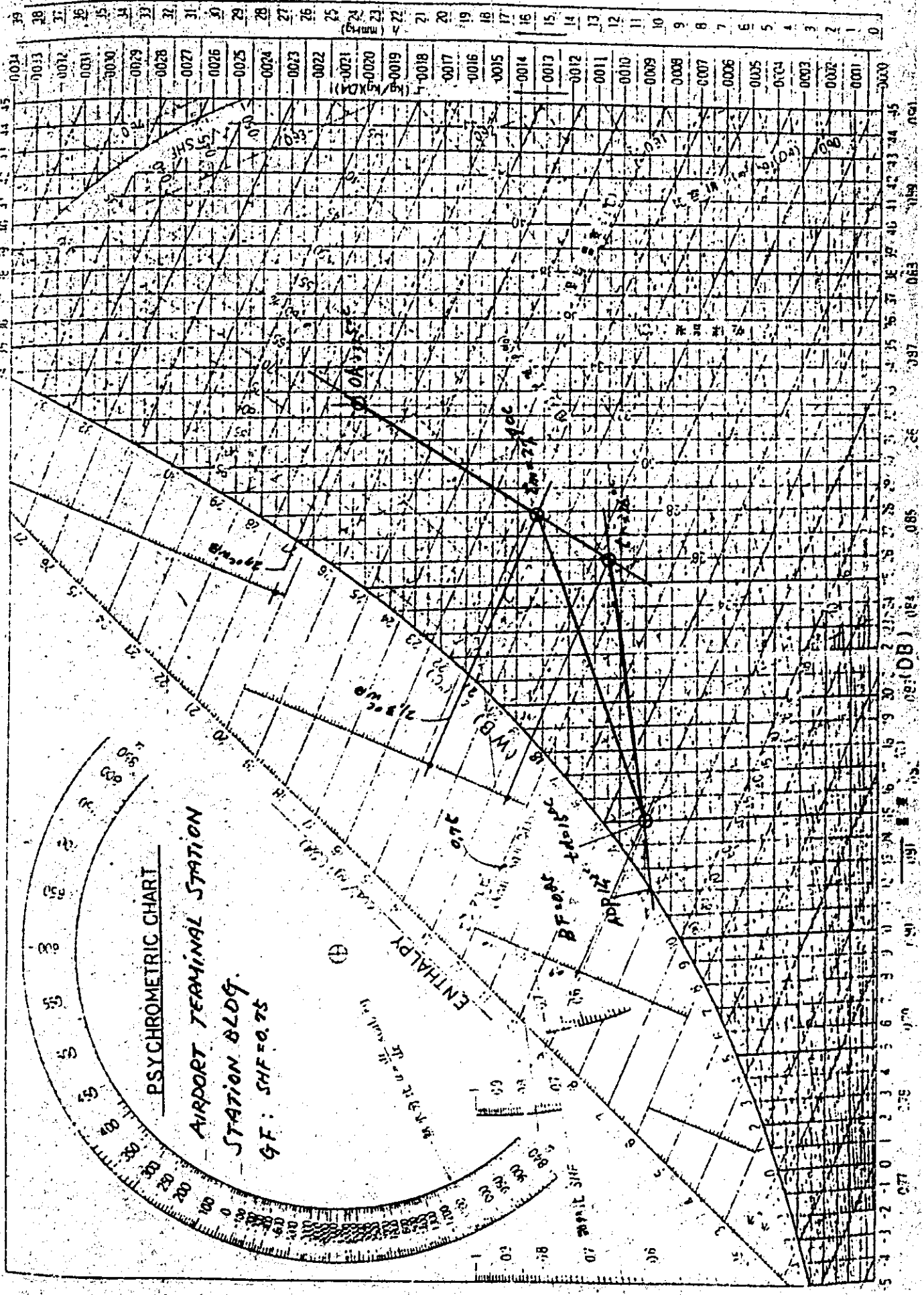
Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
E=N	Outside C.	1.45	52.3	7	8	10	706	807	1,008			
S=N		1.45	17.2	7	8	10						
E=N	Window	5.5	0.8	10.6	7.6	8.6	181	164	147			
S=N		5.5	2.3	10.6	9.6	8.6						
	Parti- m- m y	2.5	17	3.25	3.25	3.25	139	139	139			
	Window											
	Floor											
	Roof	0.86	53.1	17	19.5	18.5	777	891	845			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		
	Area (m ²)	Factor	m ²								c/h	
E=N			0.8	64	63	51	199	196	159			m ³ /h
S=N			2.3	64	63	51						m ³ /h x deg x
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss C. Total		
	Incandescent lamp			KV								
	Fluorescent lamp	20	14/2	1.07	KV		1,070	1,070	1,070			
	Meter 0.09KV - 0.37KV			KV								
	0.37KV - 1.5KV			KV								
	Gas equip. the other	0.5 x 0.60					430	430	430			
Internal Heat (Equipment) Sub. Total										Latent Heat		
	Person	53.1 x 0.2 = 11			11 x 58		638	638	638	11 x 98		528
	Infiltration	132.8 x 0.6 = 67 m ³ /h			x 6.5 deg x 0.28		122	122	122	715 x Δx		470
Heat gain Sub. Total										Supply Air Temp. Difference ° deg		
							4,162	4,457	4,558	0.0040		998
Heat gain Grand Total							5,260	5,955	5,558			
Sensible Heat Rate SHR							5,790	6,010	6,120			
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$										0.02		
										Air change		

AIRPORT TERMINAL STATION

Story: STATION BLDG. Rm. Temp. (g) °C (f) °C A.H. (g) % (f) % No. 9
 Rm. Name: DINING Rm. (V) 32 = x L 9 = = Area 228 m², H 2.5 = = Volume 81 m³

M-15

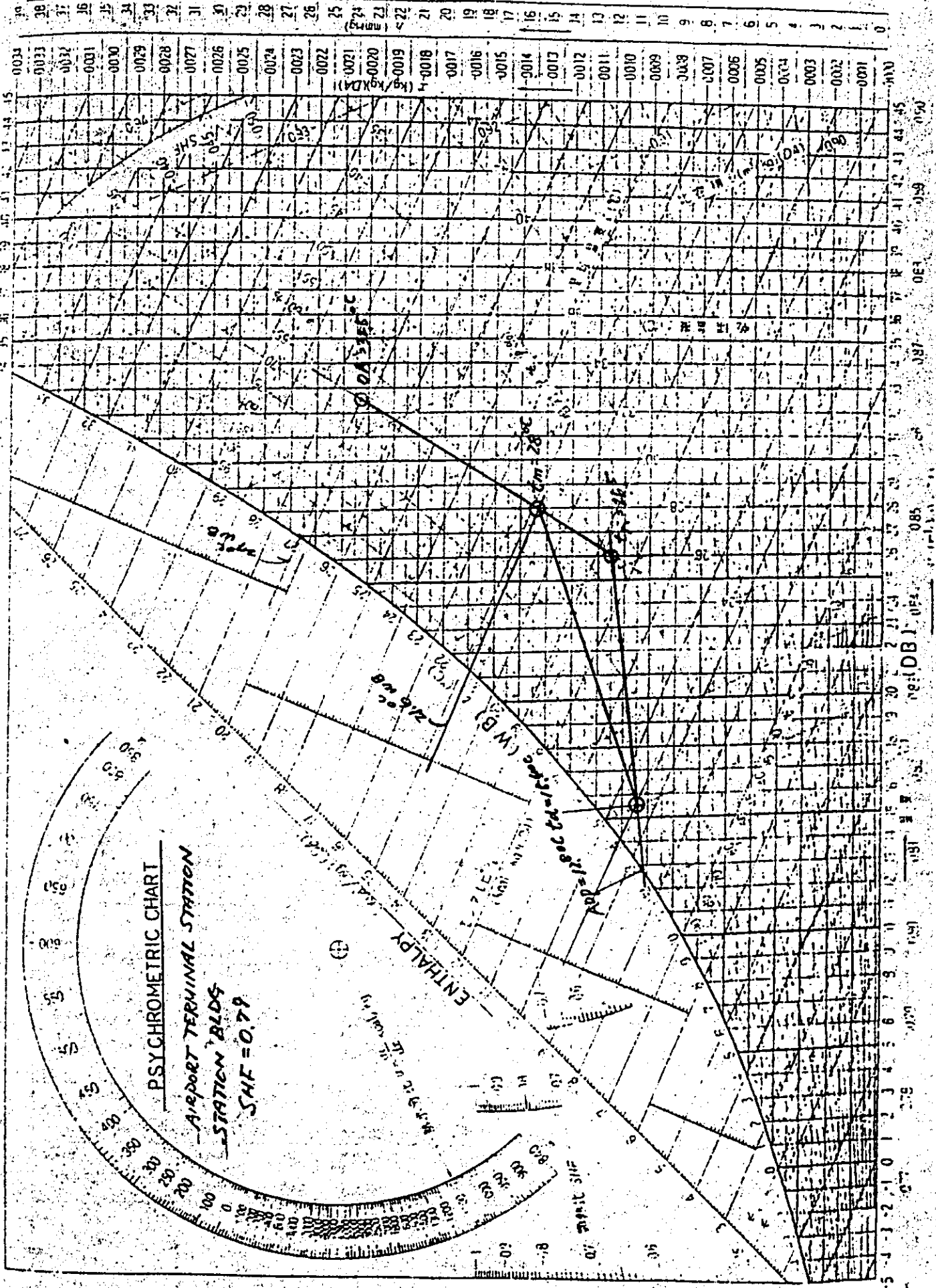
Orientation	Wall	Heat Gain (Summer)							Heat Loss (Winter)			
		K	Area m ²	Δ t (deg)			S.H. kcal/h			Δ t deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
E x N	Outside K.	1.45	4.5	7	8	10						
S x N	-	1.45	9.0	7	8	10	350	400	799			
N x N	-	1.45	20.9	7	8	10						
W x N	Window	5.5	1.6	10.6	9.6	8.6	94	85	76			
	Parti- tion	2.5	16.5	3.25	3.25	3.25	135	135	135			
	Window											
	Floor	2.0	1.5	3.25	3.25	3.25	43	43	43			
	Roof	0.66	22.4	17	19.5	18.5	477	544	516			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		
Area (m ²)	Factor									a/h		
		1.6	64	63	51	103	101	82		m ³ /h		
										Sub. T.		
Sub. Total										Factor		
Internal Heat (Equipment)										Heat Loss G. Total		
	Incandescent lamp				KV							
	Fluorescent lamp	20	N/A	0.65	KV	650	650	650				
	Meter 0.09KV - 0.37KV				KV							
	0.37KV - 1.5KV				KV							
										Supply Air Temp. Difference = deg		
										Latent Heat		
	Gas equip. the other		0.5 x 860			430	430	430				
Internal Heat (Equipment) Sub. Total												
	Person	32.9 x 0.3 = 10				10 x 50	500	500	500	10 x 25	250	250
	Infiltration	81 x 1 = 81 m ³ /h				6.5 deg x 0.28	140	140	140	7.5 x Δx	360	360
Heat gain Sub. Total												
Heat gain Grand Total							3,007			3,116 (3,159)		
Sensible Heat Rate SHR							4,920			4,560 4,610		
Required Air Volume $\frac{SH}{0.28 \times \Delta t}$										250 m ³ /h Person		
										Air change		



0.034
 0.033
 0.032
 0.031
 0.030
 0.029
 0.028
 0.027
 0.026
 0.025
 0.024
 0.023
 0.022
 0.021
 0.020
 0.019
 0.018
 0.017
 0.016
 0.015
 0.014
 0.013
 0.012
 0.011
 0.010
 0.009
 0.008
 0.007
 0.006
 0.005
 0.004
 0.003
 0.002
 0.001
 0.000

5 4 3 2 1 0
 1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44 45

0.07
 0.075
 0.08
 0.085
 0.09
 0.095
 0.1
 0.105
 0.11
 0.115
 0.12
 0.125
 0.13
 0.135
 0.14
 0.145
 0.15
 0.155
 0.16
 0.165
 0.17
 0.175
 0.18
 0.185
 0.19
 0.195
 0.2
 0.205
 0.21
 0.215
 0.22
 0.225
 0.23
 0.235
 0.24
 0.245
 0.25
 0.255
 0.26
 0.265
 0.27
 0.275
 0.28
 0.285
 0.29
 0.295
 0.3
 0.305
 0.31
 0.315
 0.32
 0.325
 0.33
 0.335
 0.34
 0.345
 0.35
 0.355
 0.36
 0.365
 0.37
 0.375
 0.38
 0.385
 0.39
 0.395
 0.4
 0.405
 0.41
 0.415
 0.42
 0.425
 0.43
 0.435
 0.44
 0.445
 0.45



LIST OF COOLING LOAD FOR AIRPORT TERMINAL STATION
STATION BLDG.

NAME OF ROOM	Cooling Load KCal/HT			SA m ³ /HT	OA		Grand TH KCal/HT	RA m ³ /HT	Remarks
	SH	LH	TH		m ³ /HT	KCal/HT			
1F									
⑤ VIP WAITING	1522 2010	609 670	2431 (2680)	297 700	5P 200				AD-200 x 2
⑥ STATION MASTER	929 1030	208 290	1192 (1320)	400	2P 50				AD-200 x 1
⑦ STATION OFFICE-1	2611 2850	1036 1146	3647 (4020)	1.000	3P 200				AD-250 x 1 AD-200 x 1
⑧ E. TICKET	2633 2900	756 570	3319 (3770)	1.000	4P 100				AD-200 x 1 SL-165 x 1000 ~ 2
Sub Total	8520	2970	(11790)	(3100)	550			2550	R/G = 300 x 600
2F									
⑨ MEETING	3821 4210	1148 1270	4968 (5430)	1500	15P 350				AD-300 x 2
⑩ POWER INSPECTION	2954 3250	557 620	3511 (3870)	1.100	4P 100				AD-250 x 2
⑪ SIGNAL TELECOM	3585 3950	901 950	3986 (4400)	1400	3P 100				AD-300 x 1 AD-200 x 1
⑫ DINING	3159 3480	1028 1140	4187 (4620)	1200	10P 250				SL-300 x 1000 ~ 2
Sub Total	14890	3980	(18370)	(5700)	630			4710	Return Duct 800 x 150 R/G = 1000 x 600
Grand Total	23710	6950	(30160)	(8300)	(1380)				
Require Air Volume = $\frac{SH}{0.28 \times \Delta T} = \frac{23710}{0.28 \times (26-15.5)} = 7990 \text{ (m}^3/\text{hr)}$ (8300 --- O.K.)									
Selection of Air Con. Safety Factor 1.5% (Kcal/hr) $7990 \times 1.15 = 9188.5 \approx 9200 \approx 16 \text{ RT} \rightarrow \text{PAC-1}$ --- O.K.									
⑬ REST	4558 6920	998 1100	5556 (6120)		11P 280				WAC-2 4300 Kcal/hr x 2

AIRPORT TERMINAL STATION

M-20

Story: BOOKING BLDG. No. 1
 No. Name: 10 TICKETS STAGES-2 No. (W) 6 x L 7 = Area 62 m², H 2.5 = Volume 155 m³
 No. Temp. (s) °C (w) °C, R.H. (s) % (w) %

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
N	Outside E.	1.25	13.4	7	8	10						
W=N	"	1.25	17.5	7	8	10	402	460	575			
S=N	"	1.25	8.7	7	8	10						
N	Window	5.5	1.6	10.6	9.6	8.6	461	418	379			
S=N	"	5.5	6.3	10.6	9.6	8.6						
	Parti- tion											
	Window											
	Floor											
	Roof	0.86	62	17	19.5	18.5	907	1040	987			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		c/h
	Area (m ²)	Factor	m ²									m ³ /h
N			1.6	69	63	51	506	498	403			m ³ /h x deg x
S=N			6.3	69	63	51						
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Factor		
Internal Heat (Equipment) Sub. Total										Heat Loss C. Total		
	Incandescent lamp			KV								
	Fluorescent lamp	20	4/6	1.29	KV	1000	1240	1240	1240			
	Motor 0.09KV - 0.37KV			KV								
	0.37KV - 1.5KV			KV								
										Supply Air Temp. Difference deg		
										Latent Heat		
	Gas equip. the other		0.5 x 860				430	430	430			
Internal Heat (Equipment) Sub. Total										Person		
	Person				4 x 65		260	260	260	4 x 65		260
	Infiltration	155 x 1 =	155 m ³ /h				283	283	283	715 x Δt		1087
Heat gain Sub. Total										Heat Loss C. Total		
Heat gain Grand Total										Factor		
Sensible Heat Rate SHR										Air change		
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$												

AIRPORT TERMINAL STATION

M-21

STORY BOOKING BLDG. Rm. Temp. (°C) 21.0 °C. R.H. (%) 75 %
 STATION OFFICE-2 Rm. (W) 2.6 m x L 5 m = Area 13 m², H 2.5 m = Volume 45 m³ No. 2

Orientation	Wall	Heat Gain (Summer)							Heat Loss (Winter)		
		K	Area m ²	Δ t (deg)			S.H. kcal/h			Δ t deg	Factor
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃		
N	Outside W.	1.95	0.6	7	8	10	88	100	125		
N	Window	5.5	3.9	10.6	9.6	8.6	228	206	185		
	Parti- tion W	2.5	9	4.5/2	3.25	3.5	74	74	74		
	Window										
	Floor										
	Roof	0.86	18	17	19.5	18.5	269	302	287		
Heat gain Sub. Total										Sub. Total	
Solar Gain-Glass										Infiltration c/h	
	Area (m ²)	Factor	m ²								m ³ /h
N			3.9	62	63	51	250	246	199		m ³ /h x deg x
Sub. Total										Sub. T.	
Internal Heat (Equipment)										Factor	
Internal Heat (Equipment) Sub. Total										Heat Loss C. Total	
	Incandescent lamp			KV							
	Fluorescent lamp	20	14/m ²	0.36 KV			360	360	360		
	Meter 0.09KV - 0.37KV			KV							
	0.37KV - 1.5KV			KV							
										Supply Air Temp. Difference ° deg	
										Latent Heat	
	Gas equip. the other										
Internal Heat (Equipment) Sub. Total										Factor	
	Person			2 x 58			116	116	116	2 x 48	96
	Infiltration	45 x 0.5 = 22.5	22.5 m ³ /h		2.5 deg x 0.28		42	42	42	715 x Δx	162
Heat gain Sub. Total							1,022	1,446	1,388	1,098	258
Heat gain Grand Total							1,680	1,707	1,646	D.A. 25	m ³ /h Person
Sensible Heat Rate SHR							1,850	1,880	1,820	50	m ³ /h
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$								0.85			m ³ /h
										Air change	

AIRPORT TERMINAL STATION

11-22

Story BOOKING BLDG. Rm. Temp. (g) °C (u) °C R.H. (g) %
 (2) BUS TICKET Rm. (V) $3.5 \text{ m} \times 6.5 \text{ m} = \text{Area}$ 22.8 m^2 H $2.5 \text{ m} = \text{Volume}$ 57 m^3
 Rm. Name

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
S=N	Outside V.	1.45	12	7	8	10	122	180	179			
S=N	Window	5.5	4.3	10.6	9.6	8.6	251	228	204			
	Parti-tion V											
	Window											
	Floor											
	Roof	0.86	22.8	17	17.5	18.5	339	383	263			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor	m ²								m ³ /h	
S=N			7.3	64	63	51	276	271	220		m ³ /h x deg x	
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss G. Total		
	Incandescent lamp			KV								
	Fluorescent lamp	20	24/m ²	0.96 KV			460	460	460			
	Meter	0.09KW - 0.37KW		KV								
		0.37KW - 1.5KW		KV								
										Supply Air Temp. Difference ° deg		
										Latent Heat		
	Gas equip. the other		0.5 x 860				430	430	430			
Internal Heat (Equipment) Sub. Total												
	Person				3 x 65		195	195	195	3 x 65		195
	Infiltration	57 x 1 = 57 m ³ /h					104	104	104	7.25 x Δt		400
Heat gain Sub. Total							2,172	(2,273)	2,050	6,000		(595)
Heat gain Grand Total							2,267	(2,806)	2,695	O.A. 25		m ³ /h Person
Sensible Heat Rate SHR								0.72		25		m ³ /h
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$												m ³ /h
										Air change		

AIRPORT TERMINAL STATION

STORY BOOKING BLDG.

Rm. Temp. (°F)

(°C)

R.H. (%)

(%)

M-23

Rm. No. SECURITY

Rm. (W) 35 x L 36 = Area

126 m² H

= Volume

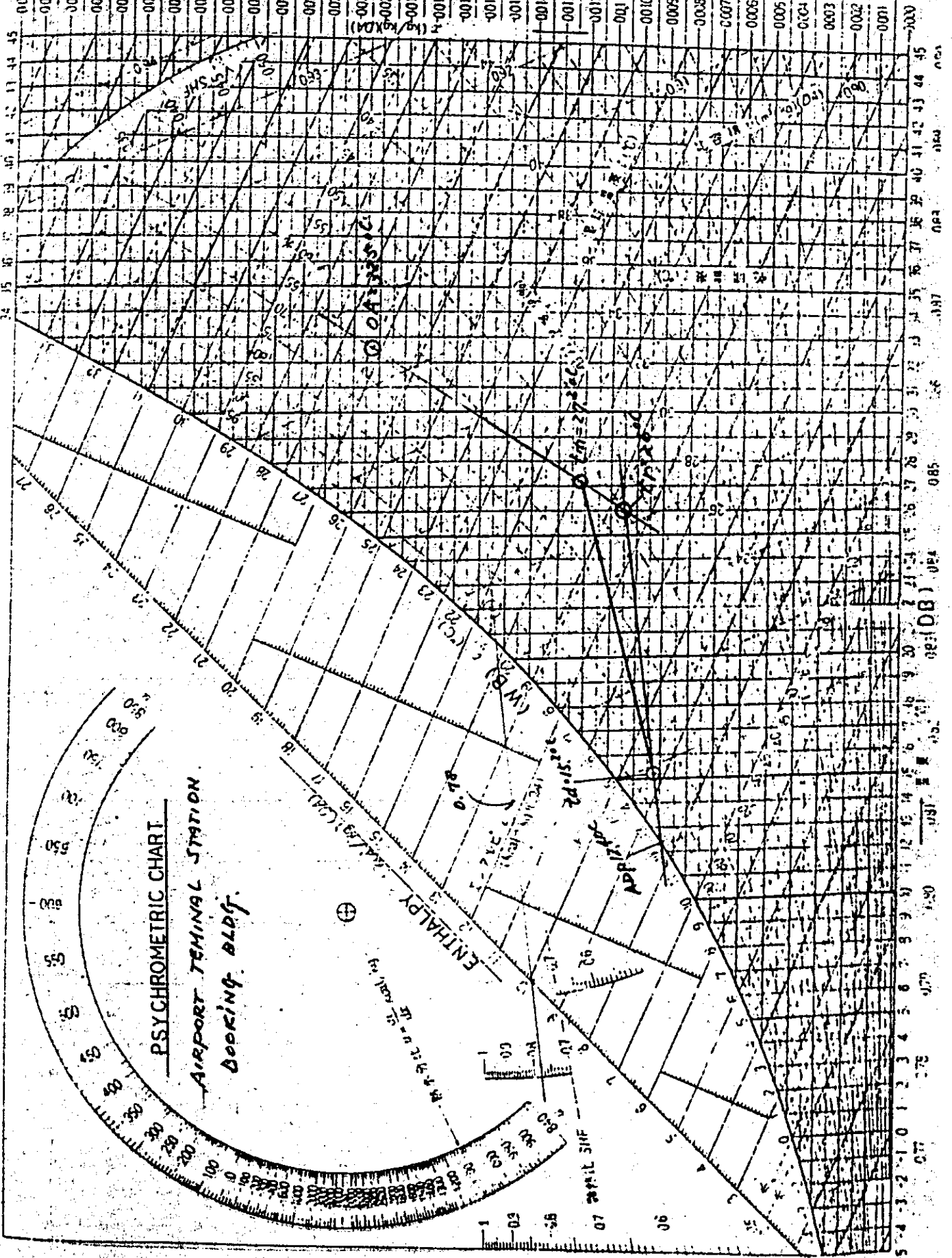
315 m³

No. 4

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
E=N	Outside v.	1.25	7.7	8	10		165	188	205			
S=N	"	1.25	9.1									
S=N	Window	5.5	1.6	10.6	9.6	8.6	94	85	76			
	Parti- -ment	2.5	9.1	3.25	2.25	3.25	74	79	74			
	Window											
	Floor											
	Roof	0.86	12.6	17	19.5	18.5	185	212	201			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor	m ²									
S=N			1.6	64	62	51	103	101	82			m ³ /h x deg x
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss C. Total		
	Incandescent lamp			KV								
	Fluorescent lamp	20	4/07	0.26	KV		260	260	260			
	Water 0.09KV - 0.37KV			KV								
	0.37KV - 1.5KV			KV								
Gas equip. the other										Supply Air Temp. Difference ° deg		
Internal Heat (Equipment) Sub. Total										Latent Heat		
	Person				4 x 50		232	232	232	4 x 40		192
	Infiltration	315 x 1	32 m ³ /h		x 6.5 deg x 0.28		59	59	59	7.5 x Δt		285
Heat gain Sub. Total							1172	1211	1219	9.028		417
Heat gain Grand Total							1539	1628	1676	O.A. Δt		m ³ /h Person
Sensible Heat Rate SHH							1750	1800	1800	100		m ³ /h
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$									0.75			m ³ /h m ²
										Air change		

LIST OF COOLING LOAD FOR AIRPORT TERMINAL STATION

NAME OF ROOM	Cooling Load KCal/HT			SA m ³ /HT	OA		Grand TH KCal/HT	RA m ³ /HT	Remarks
	SH	LH	TH		m ³ /HT	KCal/HT			
BOOKING BLDG. ...	2 BLDG.								(59) ± R m ³ /hr
(45) STATION OFFICE-2	1996 (1600)	258 (290)	1704	302 600	21 50			550	AD-250x1 (8=600)
(46) TICKET SALES-2	4629 5100	1397 1490	5976	1700	91 100			1600	AD-300x1 (700) SL-165x100 (500x2) -2
(47) BUS TICKET	2211 2990	595 660	2806	900	31 80			820	AD-250x1 (500) AD-200x1 (400)
(48) SECURITY	1219 1350	417 460	1636	500	41 100			900	AD-250 (500)
TOTAL	10990	2900	13390	3700	330	12800 2920		3470	
SHF = 0.78									
<p>Required Air Volume = $\frac{SH}{0.28 \times \Delta T} = \frac{10,990}{0.28 \times (26-15)} = 3,480 \frac{m^3}{hr}$ (at 100%) $< 3,700 \frac{m^3}{hr}$ O.K.</p>									
<p>Selection of Air-Con. Safety Factor 10% G. TH = 16,310 KCal/hr. $16,310 \times 1.1 = 18,000 \frac{KCal}{hr}$ → AC-2</p>									
"AC-2" Packaged Indoor Unit : Comp. 6.9RTx5.5KW Fan 3.700 m ³ /hr x 2.75KW x 3φ									
Condensing Outdoor Unit : Fan 0.36KW x 3φ									



AIRPORT TERMINAL STATION

11-27

Story SIGNAL No. Temp. (s) 5.9 °C (w) 5.9 °F R.H. (s) 70 %
 No. Name CONTROL Area (w) 5.9 x L 6.8 = Area 40.12 m² H 2.9 = Volume 117.35 m³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
N	Outside W.	5.4	6.6	7 x 1.3	8 x 1.3	10 x 1.3	175	200	250			
E	"	5.4	7.2	18 "	19 "	18 "	540	570	540			
S	"	5.4	6.0	17 "	20 "	22 "	225	500	550			
W	"	5.4	0.96	10 "	12 "	14 "	40	48	56			
N	Window	5.5	10.56	10.6	9.6	8.6						
E	"	5.5	16.5	10.6	9.6	8.6						
S	"	5.5	0.85	10.6	9.6	8.6	2726	2987	2228			
W	"	5.5	1.06	10.6	9.6	8.6						
	Parti-tion y.	2.5	19.5	6.5/2	6.5/2	6.5/2	159	159	159			
	Window											
	Floor	2.0	38.8	6.5/2	6.5/2	6.5/2	290	290	290			
	Roof	0.56 x 0.83	38.8	33.1	38.1	36.1	370	1001	949			
Heat gain-Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		c/h
	Area (m ²)	Factor	m ²								m ³ /h	
N	6	0.56	3.4	87	63	51	218	205	174		m ³ /h x deg x	
E	7.2	0.56	4.1	167	133	98	685	546	402			
S	6	0.56	3.4	159	119	85	524	405	289			
W	0.96	0.56	0.6	210	322	236	138	194	142			
Sub. Total										Sub. Total		
Internal Heat (Equipment)										Heat Loss G. Total		
	Incandescent lamp											
	Fluorescent lamp	20	14/2	0.74			240	740	740			
	Meter, 0.09KV - 0.37KV											
	0.37KV - 1.5KV											
										Supply Air Temp. Difference ° deg		
										Latent Heat		
	Gas-equip. and other											
Internal Heat (Equipment) Sub. Total												
	Person				3 x 58		174	174	174	3 x 60	184	
	Infiltration	142 x 0.56	72	m ³ /h	1.65 deg x 0.28		132	132	132	715 x Δx	505	
Heat Gain Sub. Total												
										O.A. 25		m ³ /h Person
Heat Gain Grand Total										75		m ³ /h
Possible Heat Refs. SHR												m ³ /h.m ²
Required Air Volume = $\frac{SH}{0.28 \Delta t}$												m ³ /h
										Air change		

AIRPORT TERMINAL STATION

11-25

STORY SIGNAL CABIN

Rm. Temp. (s)

°C (w)

°C, F.P. (s)

°C (w)

No.

Rm. Name

INSPECTORS

Rm. (V) 5

x L 6

a = Area

20 m²

H 2.9

= Values

1.7 m³

Orientation	Wall	Heat Gain (Summer)								Heat Loss (Winter)		
		K	Area m ²	Δ t (deg)			S.H. kcal/h			Δ t deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
N	Outside v.	2.2	15.0	7x1.3	8x2.3	10x1.3	237	500	624			
S	"	2.2	13.5	17 "	20 "	22 "	255	1,124	1,236			
W	"	2.2	21.9	10 "	12 "	14 "	911	1,093	1,276			
S	Window	5.5	1.5	10.6	9.6	8.6	175	175	175			
W	"	5.5	1.5	10.6	9.6	8.6						
	Parti- tion v.	2.5	23.2	3.25	3.25	3.25	191	191	191			
	Window											
	Floor	2.0	30	3.25	3.25	3.25	195	195	195			
	Roof	0.86x0.86	30	32.1	32.1	36.1	709	816	774			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		
	Area (m ²)	Factor	m ²							c/b m ³ /h		
S	1.5	0.56	0.84	154	119	85	130	100	72	m ³ /h x deg x		
W	15	0.56	8.4	230	322	236	194	271	199			
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss		
	Incandescent lamp									G. Total		
	Fluorescent lamp	20 W/m ²	0.6 KV				600	600	600			
	Meter	0.09KV - 0.37KV										
		0.37KV - 1.5KV								Supply Air Temp. Difference ° deg		
Latent Heat												
	Gas equip. the other											
Internal Heat (Equipment) Sub. Total												
	Person			3 x 58			174	174	174	3 x 48		174
	Infiltration	117x0.5 = 59 m ³ /h					108	108	108	715r, Δx		108
HEAT GAIN Sub. Total							4,779	5,275	5,624	0.0098		5,581
Heat gain Grand Total							5,237	5,833	6,182	O.A. 25		m ³ /h Person
Available Heat Rate SH							5,820	6,320	6,810	75		m ³ /h
Required Air Volume = $\frac{SH}{0.28 \Delta T}$												m ³ /h
Air change												

KOTA INTAN STATION

14-32

Story **1st** Am. Temp. (z) **26 °C (w)** °C. R.H. (a) **50 % (v)** No. **1**
 Name **STATION OFFICE** m. (V) = h.L. = Area **89.5 m² H** = Volume **392.5 m³**

Orientation	Wall	Heat Gain (Summer)							Heat Loss (Winter)			
		K	Area m ²	Δ t (deg)			S.H. kcal/h			Δ t deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
S	Outside E.	1.45	30	17	20	22	740	870	957			
W=N		1.45	58.8	7	8	10	597	683	853			
W=N	Window	5.5	7.2	10.6	9.6	8.6	420	381	391			
N	Parti- tion	2.5	96	6 1/2	6 1/2	6 1/2	744	744	744			
	Window											
	Floor											
	Roof	0.99	53	19	21	22	460	509	532			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		
	Area (m ²)	Factor	m ²							c/h		
W=N			7.2	64	63	51	461	454	368	m ³ /h x deg x		
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss		
Incandescent lamp										G. Total		
Fluorescent lamp 80W 10 W/m 1.6 kv 1000 kcal/h												
Meter 0.09kv - 0.37kv												
0.37kv - 1.5kv										Supply Air		
Other 1.8 kw 8.60°										Temp. Difference		
Gas equip. the other										° deg		
Internal Heat (Equipment) Sub. Total										Latent Heat		
Person 12 x 65										12 x 65		
Infiltration 392.5 m ³ /h x 65 deg x 0.28										715 715 715 715 x 0.0035		
Heat gain Sub. Total										2610		
Heat gain Grand Total										3390		
Possible Heat Rate SH										11,197 11,916 11,570		
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$										O.A. 25 m ³ /h Perap		
										300 m ³ /h		
										m ³ /h, m ²		
										m ³ /h		
										Air change		

KOTA INTAN STATION

14-33

Story 2E Rm. Temp. (s) °C (w) °C R.H. (s) % (w) %
 Rm. Name DINING Rm. (V) 10.2 m x L 5 m = Area 51 m² H 2.7 m = Volume 137.3 m³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area m ²	Δ t (deg)			S.H. kcal/h			Δ t deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
S	Outside v.	1.25	9.0	17	20	22	230	270	297			
E	"	1.25	20.0	18	19	18	593	574	543			
S	Window	3.5	2.2	10.6	9.6	8.6	182	175	157			
E	"	3.5	1.7	10.6	9.6	8.6						
N	Parti- tion	2.5	11.5	6.5/2	2.25	2.25	94	94	94			
	Window											
	Floor											
	Roof	0.44	57	19	21	22	427	472	494			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor										
S			2.2	159	119	83	339	262	187			
E			1.1	167	133	98	187	147	108			
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss G. Total		
	Incandescent lamp											
	Fluorescent lamp	51	204/24	1		1,000	1,100	1,100	1,100			
	Motor 0:09KV - 0:37KV											
	0:37KV - 1:5KV											
	Other			3 KW		860	2580	2580	2580			
	Gas equip. - the other											
Internal Heat (Equipment) Sub. Total										Latent Heat		
	Person				15 x 56		870	870	870	15 x 48		770
	Infiltration	30	137.3				1255	1255	1255	7.25 x Δt		5,214
Heat gain Sub. Total										Grand Total		
							7917	7872	7785	0.0088		5,214
							12,899	12,806	12,719	0.1		
							15,140	15,190	15,100			
Sensible Heat Rise, SHR										Air change		
							0.58					
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$												

KITA INTAN STATION

M-34

Story 1/F No. Temp. (a) °C (u) °C R.H. (e) % (u) =
 No. Name STATION MASTER No. (V) 5 = x L f = = Area 25 = 2 H 0.5 = = Volume 62.5 = 3

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	k	Area	Δt (deg)			S.H. kcal/h			Δt	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
S	Outside. C.	1.75	10.3	17	20	22	254	299	329			
S	Window	5.5	2.2	7.2	10.6	8.6	88	129	105			
	Factor											
	Window											
	Floor											
	Roof											
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		
	Area (m ²)	Factor	cm ²									c/h
S			2.2	154	119	85	285	298	213			m ³ /h x deg =
Sub. Total										Sub. Total		
Internal Heat (Equipment)										Heat Loss G. Total		
	Incandescent lamp											
	Fluorescent lamp	20	47/m ²	0.5 KW	1000		500	500	500			
	Water 0.09KW - 0.37KW											
	0.37KW - 1.5KW											
	Gas equip. the other											
Internal Heat (Equipment) Sub. Total										Latent Heat		
	Person			2 = 58			116	116	116	2 x 45		96
	Infiltration	62.5 x 0.5 = 31.25					59	59	59	7.5 x 0.5 = 3.75		225
Heat gain Sub. Total										Heat Loss G. Total		
							1902	1401	1322			
Heat gain Grand Total							1722	1722	1693	G.A. 25 = 3/h Person		
Sensible Heat Rate SHR							1900	1900	1810	50 = 3/h		
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$							0.82	0.82		= 3/h		
										Air change		

KOTA JINTAN STATION

M-35

STORY 1/F No. Temu. (s) °C (w) °C P.M. (s) (w) No. 4
 No. MEETING No. (w) 3 x L 5 m = Area 25 m² H 2.5 m = Volume 62.5 m³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	L	Area	Δt (deg)			S.H. kcal/h			Δt deg	Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
E-11	Outside W.	1.75	25	7	8	10	254	290	363			
	Window											
N	Parti-tion	75	125	225	225	225	102	102	102			
	Window											
	Floor											
	Roof	0.44	25	19	21	22	214	237	298			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration		
Area (m ²)		Factor		m ²						c/h		
										m ³ /h		
										m ³ /h x deg x		
										Sub. T.		
Sub. Total										Factor		
Internal Heat (Equipment)										Heat loss G. Total		
Incandescent lamp												
Fluorescent lamp 20 $\frac{100}{100}$ 105 KW										1,000 500 500 500		
Water 0.03KW - 0.37KW												
0.37KW - 1.5KW												
										Supply Air Temp. Difference ° deg		
Gas equip. etc. other										Latent Heat		
Internal Heat (Equipment) Sub. Total												
Person 25 $\frac{100}{100}$ 105 KW = 13.75 KW										113 x 50 754 754 754 13 x 48 624		
Infiltration 32 m ³ /h										3.65 deg x 0.20 59 59 59 7.5 x 25 225		
Heat gain Sub. Total										1853 1972 2076 1849		
Heat gain Grand Total										2732 2991 2879 D.A.		
Sensible Heat Rate SH										2010 2640 2730		
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$										0.90 m ³ /h		
										Air change		

KOTA INTAN STATION

M-26

Story 1/F No. Temp. (s) °C(w) °C (R.H. (s) % (w) =
 No. Name SECURITY No. (V) 38 m x L 3.0 m Area 19 m² H 2.5 m Volume 47.5 m³ No. 5

Heat Gain (Summer)										Heat Loss (Winter)	
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃		
S E-N	Outside C.	1.43	11.9	17	20	22	282	231	268		
		1.43	7.5	7	8	10	77	81	109		
S	Window	3.5	11	10	9	6	65	59	52		
	Parti- -roovy										
	Window										
	Floor										
	Roof	0.64	17	19	21	22	159	176	184		
Heat gain Sub-Total										Sub-Total	
Solar Gain-Glass										Infiltration	
	Area (m ²)	Factor	m ²								
S			11	154	119	85	170	121	94		
Sub-Total										Sub-Total	
Internal Heat (Equipment)										Heat loss G. Total	
	Independent Lamp										
	Fluorescent Lamp	20	1/100	0.20 Xv			1000	250	250	250	
	Motor: 0.02KV - 0.37KV										
	0.37KV - 1.5KV										
	Gas equip.										
	the other										
Internal Heat (Equipment) Sub-Total										Supply Air Temp. Difference ° deg	
	Person			4 x 58			232	232	232	4 x 40	192
	Infiltration	47.5 x 0.5 = 24 m ³ /h		1.65 deg x 0.20			49	49	49	7.5 x 1.99	169
Heat gain Sub-Total										Latent heat	
							1809	1880	(1869)		(367)
Heat gain Grand Total										Air change	
				Factor 1.1			1990	1801	(1860)	O.A.	3/h Person
Sensible Heat Rate SHR										3/h	
							1950	1990	2010		3/h x 2
Required Air Volume = $\frac{SH}{0.20 \times 1.2}$										3/h	
Provided Air Volume = $\frac{SH}{0.20 \times 1.2}$										Air change	

RETAINING STATION

M-37

Story 2nd No. Temp. (a) (b) (c) F.H. (a) (b) (c) No. 6
 No. Name PROJECT No. (W) 5 m x L 72 m Area 26 m² H 4.5 m Volume 80 m³

Heat Gain (Summer)										Heat Loss (Winter)	
Orientation	Wall	K	Area m ²	Δt (deg)			S.H. kcal/h			Factor	Heat loss kcal/h
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃		
S	Outside c.	1.25	10 ³	17	20	22	259	294	329		
W	"	1.25	16 ³	10	12	17	237	288	331		
S	Window	1.5	12 ²	10.6	9.6	8.6	129	117	105		
	Parti- tion	1.25	31 ³	3.25	3.25	3.25	255	255	255		
	Window										
	Floor	1.20	36	3.25	3.25	3.25	239	239	239		
	Roof	0.74	36	19	21	22	301	333	349		
Heat gain Sub. Total										Sub. Total	
Solar Gain-Glass										Infiltration	
	Area (m ²)	Factor	m ²							c/h	
S			22	152	119	85	339	262	187	m ³ /h x deg x	
Sub. Total										Sub. T.	
Internal Heat (Equipment)										Heat Loss G. Total	
	Incandescent lamp										
	Fluorescent lamp	20	1/17	0.72 kW	1000		720	720	720		
	Motor 0.09KW - 0.37KW										
	0.37KW - 1.5KW										
Latent Heat										Supply Air Temp. Difference	
	Gas equip.										
	the other			1.5m x 860			860	260	860		
Internal Heat (Equipment) Sub. Total										Latent Heat	
	Person	20	28.3	11 person	11 x 50 x 0.5		511	511	511	11 x 20 x 0.5 x 2.5	
	Infiltration	90 x 0.5	45 m ³ /h	265 deg x 0.28			82	82	82	715 x 0.28 x 0.28	516
Heat gain Sub. Total										Heat gain Grand Total	
Heat gain Sub. Total										Sensible Heat Rate SHR	
Heat gain Sub. Total										Required Air Volume	
Heat gain Sub. Total										Air change	

KOTA INTAN STATION

M-38

Story: 2F No. Term. (g) P.H. (g) No. 7
 Rm. Name: TRAIN CONTROL No. (V): 36 Area: 17 m² H: 2.5 m x Volume: 42.5 m³

Heat Gain (Summer)										Heat Loss (Winter)		
Orientation	Wall	K	Area	Δt (deg)			S.H. kcal/h			Factor	Heat loss kcal/h	
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃			
S	Outside	1.85	10.2	17	20	22	252	296	326			
E-N		1.85	7.4	7	8	10	71	86	108			
N		1.85	11.3	7	8	10	120	147	172			
S	Window	5.5	1.82	10.6	9.6	8.6	189	172	154			
E-N		5.5	1.82									
	Window											
	Window											
	Floor	2.0	1.7	3.25	3.4	3.5	111	111	111			
	Roof	0.66	1.7	1.7	1.95	1.85	299	286	271			
Heat gain Sub. Total										Sub. Total		
Solar Gain-Glass										Infiltration c/h		
	Area (m ²)	Factor									m ³ /h	
S			1.82	154	119	85	250	193	138		m ³ /h x deg x	
E-N		20:22	1.82	69	63	51	23	23	19			
Sub. Total										Sub. T.		
Internal Heat (Equipment)										Heat Loss C. Total		
	Incandescent lamp											
	Fluorescent lamp	20	W/ft	600	KV	1000	290	290	290			
	Meter 0.08KV - 0.37KV											
	0.37KV - 1.15KV										Supply Air Temp. Difference deg	
	Other		0.8KW			860	430	430	430		Latent heat	
	Gas equip. etc. other											
Internal Heat (Equipment) Sub. Total												
	Expon					W = 65	195	195	195	3.65	195	
	Infiltration	42.5 x 1.0 x 2.43	m ³ /h			2.05 deg x 0.28	79	79	79	7.5 x 4x	302	
Heat gain Sub. Total							2314	2243	2243	6.140	1297	
Heat gain Grand Total							2811	2870	2870	O.A.	m ³ /h Person	
Sensible Heat Rate SHR							0.82	0.83	0.83		m ³ /h	
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$												
Air change												

KOTA INTAN STATION

M-39

Storey: 2F No. Terminal: (50) SIGNAL TELECOM No. (10) 2.6 m x L 4.7 m Area: 12.2 m² H: 2.6 m Volume: 31.3 m³

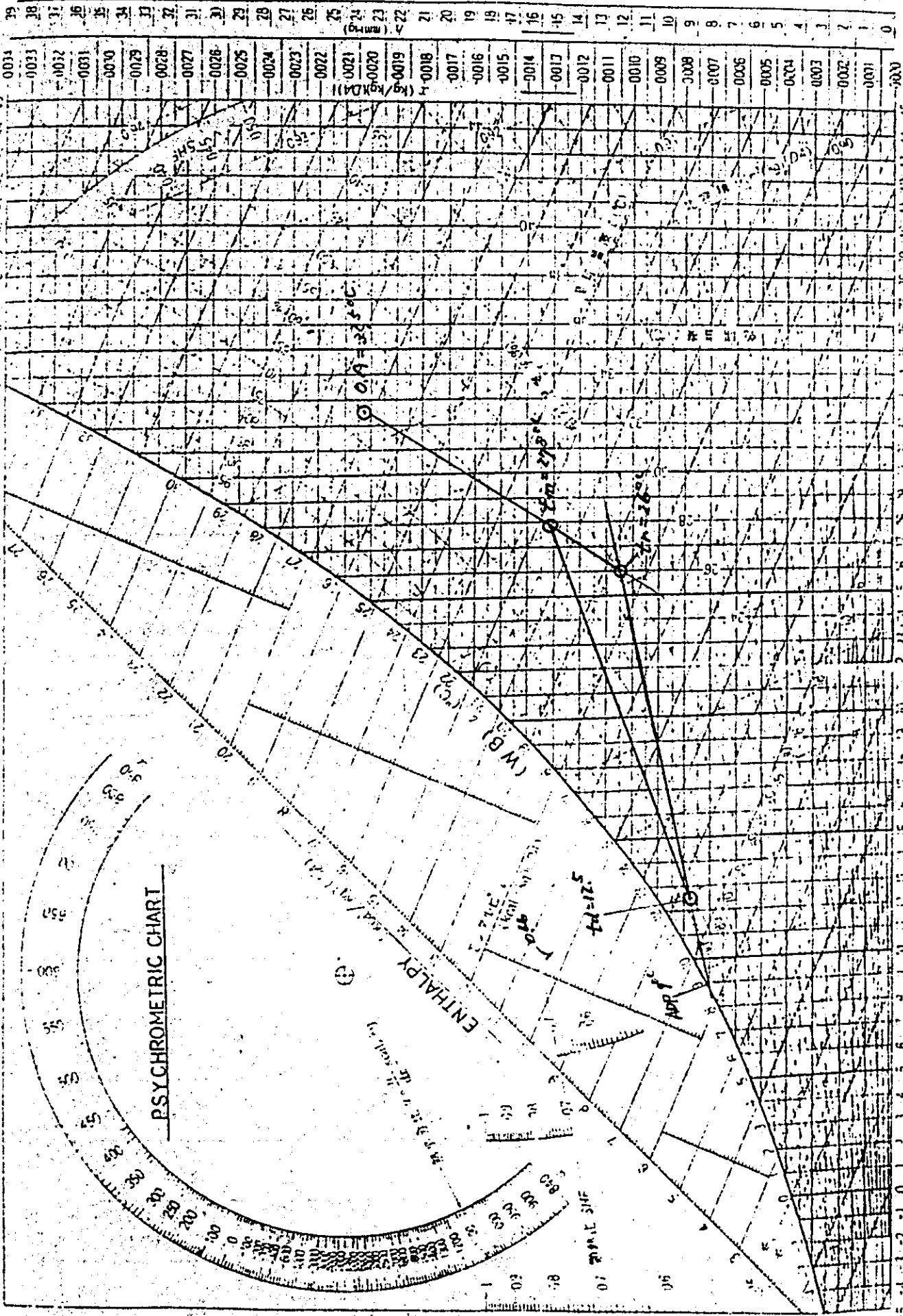
Heat Gain (Summer)										Heat Loss (Winter)			
Orientation	Wall	K	Area m ²	Δ t (deg)			S.H. kcal/h			Δ T deg	Factor	Heat loss kcal/h	
				15:00	17:00	19:00	SH ₁	SH ₂	SH ₃				
W	Outside			SOME OF TRAIN CONTROL OFFICE									
W	Window												
W	Window												
	Parti-partok												
	Window												
	Floor												
	Roof												
Heat gain Sub. Total										Sub. Total			
Solar Gain Glass										Infiltration			
	Area (m ²)	Factor	m ²								c/h		
CS											m ³ /h		
W											m ³ /h x deg x		
										Sub. T.			
										Factor			
Internal Heat (Equipment)										Heat Loss G. Total			
	Incandescent lamp		KW										
	Fluorescent lamp		KW										
	Meter 0.09KW - 0.37KW		KW										
	0.37KW - 1.5KW		KW										
										Supply Air Temp. Difference ° deg			
										Latent Heat			
	Gas equip. the other												
Internal Heat (Equipment) Sub. Total													
	Person			x						x			
	Infiltration		m ³ /h	x	deg x 0.28					x			
Heat gain Sub. Total										7.5x Δx			
										2314	(2348)	2343	(497)
Heat gain Grand Total										2811	(2845)	2840	O.A.
Heat gain Grand Total										3,100	(3,130)	3,130	m ³ /h Person
Sensible Heat Rate SHR										0.82	0.83	0.83	m ³ /h
Required Air Volume = $\frac{SH}{0.28 \times \Delta t}$													m ³ /h
Required Air Volume =													Air change

LIST OF COOLING LOAD FOR KOTA INTAN STATION

NAME OF ROOM	Cooling Load KCal/HT			SA m ³ /HT	OA		Grand TH KCal/HT	RA m ³ /HT	Remarks
	SH	LH	TH		m ³ /HT	KCal/HT			
5 STATION OFFICE	8180 (9,000)	3390 (3,730)	11,570 (12,730)	2400	120	300		1770	AD-300 x 2
8 DINING	7,915 (8,710)	5,939 (6,530)	13,854 (15,240)	2900	150	350		2020	AD-350 x 2 AD-200 x 1
9 STATION MASTER	1,902 1,550	321 (360)	1,723 1,910	500	20	50		450	AD-250 x 1
10 MEETING	2,026 (2,230)	849 (940)	2,875 (3,170)	600	130	330		270	AD-250 x 1
11 SECURITY	1,459 1,610	361 (400)	1,820 2,010	500	40	100		400	AD-200 x 1
	<u>23,100</u>	<u>11,960</u>	<u>35,060</u>	<u>6,900</u>	<u>1,160</u>	<u>10,250</u>	<u>45,310</u>	<u>2,910</u>	
SHF = 0.66									
Required Air Volume = $\frac{SH}{0.25 \times \Delta T} = \frac{23,100}{0.25(26-12.5)} = 6,120 \text{ (m}^3/\text{hr)} < 6,200 \text{ (m}^3/\text{hr)} \dots \text{O.K.}$									
Selection of Air-Cond. $\frac{GTH}{0.7} \times 1.05 = 48,000 \text{ (Kcal/hr)}$ "AC-1"									
packaged Indoor Unit: Comp. 16R Fan 6000 m ³ /hr									
Condensing Outdoor Unit: Fan									
12 REST	3,963	739	4,702 x 1.1 5,180	1100	275	1:080 (2,930)	7610 x 1.05 (8,000)		WAC-2 4,500 Kcal/hr 2 SETS
13 TRAIN CONTROL	2,348	497	2,845 x 1.1 3,130	34	75	1:080 670	2,800 x 1.05 (4,000)		WAC-2 4,000 Kcal/hr 1 SET
14 SIGNAL TELECOM	2,348	497	2,845 x 1.1 3,130	34	75	1:080 670	2,800 x 1.05 (4,000)		WAC-2 4,000 Kcal/hr 1 SET

M-41

110 70



0034	0033	0032	0031	0030	0029	0028	0027	0026	0025	0024	0023	0022	0021	0020	0019	0018	0017	0016	0015	0014	0013	0012	0011	0010	0009	0008	0007	0006	0005	0004	0003	0002	0001	0000
------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------	------

39	38	37	36	35	34	33	32	31	30	29	28	27	26	25	24	23	22	21	20	19	18	17	16	15	14	13	12	11	10	9	8	7	6	5	4	3	2	1	0
----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	----	---	---	---	---	---	---	---	---	---	---

0.000 0.001 0.002 0.003 0.004 0.005 0.006 0.007 0.008 0.009 0.010 0.011 0.012 0.013 0.014 0.015 0.016 0.017 0.018 0.019 0.020 0.021 0.022 0.023 0.024 0.025 0.026 0.027 0.028 0.029 0.030

0.000 0.001 0.002 0.003 0.004 0.005 0.006 0.007 0.008 0.009 0.010 0.011 0.012 0.013 0.014 0.015 0.016 0.017 0.018 0.019 0.020 0.021 0.022 0.023 0.024 0.025 0.026 0.027 0.028 0.029 0.030

VENTILATION LIST OF KOTA INTAN STATION

No. M-42

NAME OF ROOM	Calculation of Air Volume		Remarks
	Exhaust Air of Room	m ³ /HR	
10 PASSENGERS LAVATORY (W)	4.5 x 5 x 2.3 = 55 (m ³) 55 x 20 = 1100 (m ³ /hr)	1100 _{13.9}	HS-450x450
8 DITTO (M)	5.7 x 5 x 2.3 = 60 (m ³) 60 x 20 = 1200 (m ³ /hr)	1200 _{13.9}	HS-450x450
15 PORTER	5.2 x 5 x 2.3 = 60 (m ³) 60 x 10 = 600 (m ³ /hr)	600 ₁₀	WEF-300
6 MAINTENANCE & JANITORS	5 x 8.4 x 2.3 = 102 (m ³) 102 x 10 = 1100 (m ³ /hr)	1100 _{13.9}	WEF-400
12 STAFF LAVATORY	5 x 4 x 2.5 = 50 (m ³) 50 x 15 = 800 (m ³ /hr)	800 _{13.9}	WEF-350
13 STAFF LOCKER	5 x 4 x 2.5 = 50 (m ³) 50 x 10 = 500 (m ³ /hr)	500 _{13.9}	WEF-300
15 STORAGE - 1	4.5 x 4.3 x 2.5 = 68 (m ³) 68 x 10 = 700 (m ³ /hr)	700 _{11.7}	WEF-350
15 STORAGE - 2	108 (m ³) 108 x 10 = 1100 (m ³ /hr)	1100 _{13.9}	WEF-400
16 POWER	5.5 x 4.3 x 2.3 = 78 (m ³) 78 x 15 = 1200 (m ³ /hr)	1200 ₂₀	WEF-400
17 MECHANICAL	108 (m ³) 108 x 10 = 1100 (m ³ /hr)	1100 _{13.9}	WEF-400
18 SIGNAL TELLER DEVICE	6.2 x 7.5 x 2.2 = 104 (m ³) 104 x 15 = 1600 (m ³ /hr)	1600 _{26.7}	WEF-500
18 STORAGE - 3	3 x 6.2 x 2.3 = 62 (m ³) 62 x 10 = 700 (m ³ /hr)	700 _{11.7}	WEF-350
18 KITCHIN	3 x 3.6 x 2.3 = 25 (m ³) 25 x 30 = 800 (m ³ /hr)	800 _{13.9}	WEF-350
18 RELUTION	5 x 4 x 2.3 = 39.5 (m ³) 39.5 x 10 = 400 (m ³ /hr)	400 _{6.7}	WEF-300
18 MUSHOLA	5 x 5 x 2.3 = 92 (m ³) 92 x 15 = 1400 (m ³ /hr)	1400 _{23.4}	WEF-400
18 LAVATORY	5.4 x 3 x 2.3 = 27.9 (m ³) 27.9 x 20 = 500	500 _X	HS-300x300
① + ② + ③ = 2800 m ³ /hr (X) CEE-2 No. 2 x 2800 m ³ /hr x 15 KW x 3 φ			

LIST OF PLUMBING FIXTURES

MARK	DISCRIPTION	WATER SUPPLY (FU.)	SEWAGE (FU.)
WC-1	Asian-type Water Closet	5	4
WC-2	European-type (wash down) w/Flush Valve	10	8
UR-1	Floorstanding Urinal 380x380x920, w/Flush Valve	5	4
UR-2	Wall Hung 460x280x800, w/Flush Valve	5	4
LV-1	Wall Hung Lavatory 500x400, w/Faucet	2	1
LV-2	Round Lavatory (Counter-top) 400 ϕ , w/Faucet	2	1
SK-1	Service Sink 560x450x530, w/Faucet	3	2.5
F-1	Faucet (1/2), Wall-Type	2	1
F-2	Faucet (1/4), Wall-Type	3	2
F-3	Faucet (1/4), For Landscape	3	-
KS-2	Kitchen Sinks	4	3
FD-1	Floor Drain	-	0.5
FD-2	"	-	1
KS-1	Kitchen Sink	4	3

LIST OF PLUMBING EQUIPMENTS

SITE: AIRPORT TERMINAL STATION (1)

NAME OF ROOM & BLDG.	FIXTURE		WATER SUPPLY					SEWAGE				
	NAME	NO.	FU @	FU TOTAL	P/min	FRIC-TION	PIPE Ø	FU @	FU TOTAL	P/min	GRADE	PIPE Ø
TERMINAL BLDG.							30 mm / 1 m					
12F ABLUTION	F-1	4	5	20				4	16			
	FD-1	1	-	-				0.5	0.5			
(ABLUTION SINK)	SK-2	1	4	4				5	5			
				29	60		40 ^A		19.5			50
12F STAFF LAVATORY	W-1	1	5	5				4	4			
12F KITCHEN	UP-1	3	5	15				4	12			
	LV-1	2	2	4				1	2			
	SK-2	1	3	3				2.5	2.5			
	FD-1	3	-	-				0.5	1.5			
(COOLER DRAIN)	FD-1	1	-	-				0.5	0.5			
				27	70		40 ^A		22.5			75
12F STAFF LAVATORY	WC-1	1	5	5				4	4			
	UP-1	3	5	15				4	12			
	LV-1	2	2	4				1	2			

LIST OF PLUMBING EQUIPMENTS

SITE: AIRPORT TERMINAL STATION (2)

NAME OF ROOM & BLDG.	FIXTURE		WATER SUPPLY				SEWAGE					
	NAME	NO.	FU (Q)	FU TOTAL	P/min	FRIC -TION	PIPE Ø	FU (Q)	FU TOTAL	P/min	GRADE	PIPE Ø
	FD-1	2	-					0.5	1			
				(24)	60		40 ^A		(19)			75
VIP LAVATORY	WC-2	1	10	10				8	8			
	LV-2	1	2	2				1	1			
	FD-1	1	-	-				0.5	0.5			
STATION OFFICE DRESSING	JK-1	1	3	3				2.5	2.5			
				(15)	70		37 ^A		(12)			75

LIST OF PLUMBING EQUIPMENTS

SITE : AIRPORT TERMINAL STATION (3)

NAME OF ROOM & BLDG.	FIXTURE		WATER SUPPLY					SEWAGE				
	NAME	NO.	FU (a)	FU TOTAL	g/min	FRIC-TION	PIPE Ø	FU (b)	FU TOTAL	g/min	GRADE	PIPE Ø
BOOKING OFFICE (12)												
LAVATORY 3 & 4	WC-1	4	5	20				4	16			
	WC-2	2	10	20				8	8			
	UR-1	5	5	25				4	20			
	LV-1	5	2	10				1	5			
	SK-1	2	3	6				2.5	5			
	FD-1	4	-	-				0.5	2			
				81	150		50 ^a		56			100
LAVATORY 1 & 2	WC-1	2	5	10				4	8			
	WC-2	2	10	20				8	16			
	UR-1	3	5	15				4	12			
	LV-1	4	2	8				1	4			
	SK-1	1	3	3				2.5	2.5			
	FD-1	4	-	-				0.5	2			
				56	120		50 ^a		49.5			100
				137								

LIST OF PLUMBING EQUIPMENTS

SITE: AIRPORT TERMINAL STATION (A)

NAME OF ROOM & BLDG.	FIXTURE		WATER SUPPLY					SEWAGE				
	NAME	NO.	FU (Q)	LU TOTAL	R/min	FRIC-TION	PIPE Ø	FU (Q)	LU TOTAL	R/min	GRADE	PIPE Ø
BOOKING OFFICE (2)												
LAVATORY 3 & 4	WC-1	4	5	20				4	16			
	WC-2	2	10	20				8	16			
	UR-1	5	5	25				4	20			
	LV-1	5	2	10				1	5			
	SK-1	2	3	6				2.5	5			
	FD-1	4	-	-				0.5	2			
				(81)	150		50 ^ø		(64)			100
LAVATORY 1 & 2	WC-1	2	5	10				4	8			
	WC-2	2	10	20				8	16			
	UR-1	3	5	15				4	12			
	LV-1	2	2	8				1	4			
	SK-1	1	3	3				2.5	2.5			
	FD-1	4	-	-				0.5	2			
				56	120		50 ^ø		(44.5)			100

LIST OF PLUMBING EQUIPMENTS

SITE: AIRPORT TERMINAL STATION (S)

NAME OF ROOM & BLDG.	FIXTURE		WATER SUPPLY					SEWAGE				
	NAME	NO.	FU @	FU TOTAL	Q/min	FRIC-TION	PIPE Ø	FU @	FU TOTAL	Q/min	GRADE	PIPE Ø
SIGNAL CABIN												
TOILET	WC-1	1	5	5				4	4			
	LV-1	1	2	2				1	1			
				7	25		25 ^A		(5)			
TRASH	F-1	1	2	2				1	-			
				(2)	5		15 ^A					
Grand TOTAL				373	(780) mm ³ /hr 30		80 ^A		287		1/50 ~1/100	150
<p>ELEVATED STORAGE TANK (EST) $18/2 = 9 (m^3) \rightarrow$ Tank Capacity (12 m³) $420 \frac{l}{min} \times 0.7 = 300 \frac{l}{min} = 18 m^3/hr$ Tank Size ... (2.6 x 3.0 x 2.0 m) (FRP)</p>												
<p>WATER RESERVOIR (WR) RESERVOIR ... 3.0m x 3.0m x 2.0m (FRP) $9 m^3 \times 2 = 18 m^3$</p>												
<p>LIFT WATER PUMP (LWP) : Centrifugal Type (LWP-1) 65x50 x 500 l/min - 22 mH x 3.7 Kw x 3P</p>												

LIST OF PLUMBING EQUIPMENTS

SITE: KOTA INTAN STATION (1)

NAME OF ROOM & BLDG	FIXTURE		WATER SUPPLY					SEWAGE				
	NAME	NO.	FU (Q)	FU TOTAL	R/min	FRIC-TION	PIPE Ø	FU (Q)	FU TOTAL	R/min	GRADE	PIPE Ø
2F LAVATORY & ABLUTION	WC-1	1	5	5				4	4			
	UR-1	2	5	10				4	8			
	LV-1	2	2	4				1	2			
	SK-1	1	3	3				2.5	2.5			
	FD-1	3	-	-				0.5	1.5			
	(ABLUTION) F-2	4	3	12				2	8			
				(52) 80				(26)		1/50	100	
2F KITCHIN	SK-2	1	4	4				3	3			
	FD-1	2	-	-				0.5	1			
				4	12				4			50
1F PASSENGERS LAVATORY (M)	WC-1	5	5	25				4	20			
	UR-1	5	5	15				4	12			
	LV-1	6	2	12				1	6			
	SK-1	1	3	3				2.5	2.5			
	FD-1	4	-	-				0.5	2			

LIST OF PLUMBING EQUIPMENTS

SITE: KOTA INTAN STATION (2)

NAME OF ROOM & BLDG.	FIXTURE		WATER SUPPLY					SEWAGE				
	NAME	NO.	FU (Q)	FU TOTAL	Q /min	FRIC-TION	PIPE Ø	FU (Q)	FU TOTAL	Q /min	GRADE	PIPE Ø
				155	100		50 ^A		42.5			100
STATION OFFICE PANTRY	SK-1	1	3	3				2.5	2.5			
				3	10		20 ^A		2.5			50
STAFF LAVATORY	WC-1	2	5	10				4	8			
	UR-1	3	5	15				4	12			
	LV-1	2	2	4				1	2			
	SK-1	1	3	3				2.5	2.5			
	FD-1	2	-	-				0.5	1			
				32	85		50 ^A		23.5			100
Grand Total				128	190	(307) mmAB/m	50 ^A		100.5			100
EST	ELEVATED STORAGE TANK		$9.7/2 = 5 \text{ (m)} \rightarrow$ Tank Capacity (6 m ³)					Tank Size --- (20' x 3.0' x 10' H (FRP))				
WR	WATER RESERVOIR		$190 \text{ l/min} \times 0.85 = 9.7 \text{ m}^3/\text{hr}$					RESERVOIR $\rightarrow 20' \times 3.0' \times 2.0' \text{ H (FRP)} \quad 12 \text{ m}^3$				
LWP	LIFT PUMP (Centrifugal Type)		WATER $50 \times 90' \times 200 \text{ l/min} \times 29' \text{ H} \times 2.2 \text{ kW} \times 3 \text{ P}$					LWP-25				

SELECTION OF SEPTIC-TANK & PERCORATION PIPE FOR KOTA INTAN STATION

1. SEPTIC-TANK

SOIL : NO. OF FU = 64, $Q_0 = 130 \text{ l/min}$

SIZE OF SEPTIC-TANK

$$\text{WHERE } Q_1 = (Q_0 \times 60 \times 2) \times \frac{1}{1,000} \doteq 15.6 \text{ (m}^3\text{)}$$

$$Q_2 = Q_1 \times 1.2 \doteq 18.8 \text{ (")}$$

$$2,400^W \times 6,000^L \times 1,700^H \text{ (EFFECTIVE } 1,300^H\text{)}$$

2. LENGTH OF PERCORATION PIPE (L)

$$\text{WHERE } Q_1 + Q_2 = 15.6 + 7.8 = 23.4 \text{ (m}^3\text{)}$$

$$L = 23.4 \times 83.3 \div 60 \doteq 33 \text{ (m)} \dots 100^\phi$$

§§ 16. LIGHT INTENSITIES (LUX)

TABLE OF CONTENTS

	PAGE
[1] AIRPORT TERMINAL STATION TERMINAL BUILDING	3
[2] BOOKING OFFICE	4
[3] SIGNAL CABIN A	5
[4] SIGNAL CABIN B	6
[5] KOTA INTAN STATION	7
[6] AIRPORT TERMINAL STATION STATION PLAZA	8
[7] KOTA INTAN STATION STATION PLAZA	10

NEW RAILWAY FOR CENKARENG AIRPORT / INDONESIA

LIGHTING AREA	LUM	W	Y	FIXTURE NO.	NO. (Calc)	NO. LUM/Calc	MARKS
I21 BOOKING OFFICE							
Train control office	300	4.8	3.3	F3, FL4012	2.6	340	
Station office 2	300	4.8	3.3	F3, FL4012	2.6	340	
Ticket sales 2	300	5.8	6.8	F3, FL4012	5.6	320	
Bus ticket sales	300	3.3	3.3	F3, FL4012	2.3	265	
Waiting room	150	2.8	3.3	F3, FL4012	1.8	156	
Security	300	3.3	3.3	F3, FL4012	2.3	265	
Porter	200	6.8	2.3	F3, FL4012	2.2	277	
Lavatory 1	100	4.8	2.8	F2, FL4011	1.7	119	Water proof
Lavatory 2	100	4.8	2.3	F2, FL4011	2.0	101	Water proof
Lavatory 3	100	6.8	2.8	F2, FL4011	2.4	126	Water proof
Lavatory 4	100	6.8	3.8	F2, FL4011	2.9	104	Water proof
Storage 1	50	3.3	1.8	F2, FL4011	0.5	102	
Storage 2	50	2.8	1.8	F2, FL4011	0.4	120	
Train platform	100	8.0	100.0	F6, FL11011	43.4	110	Water proof
Bus platform	100	8.0	100.0	F6, FL11011	43.4	110	Water proof

NEW RAILWAY FOR CENKARENG AIRPORT / INDONESIA

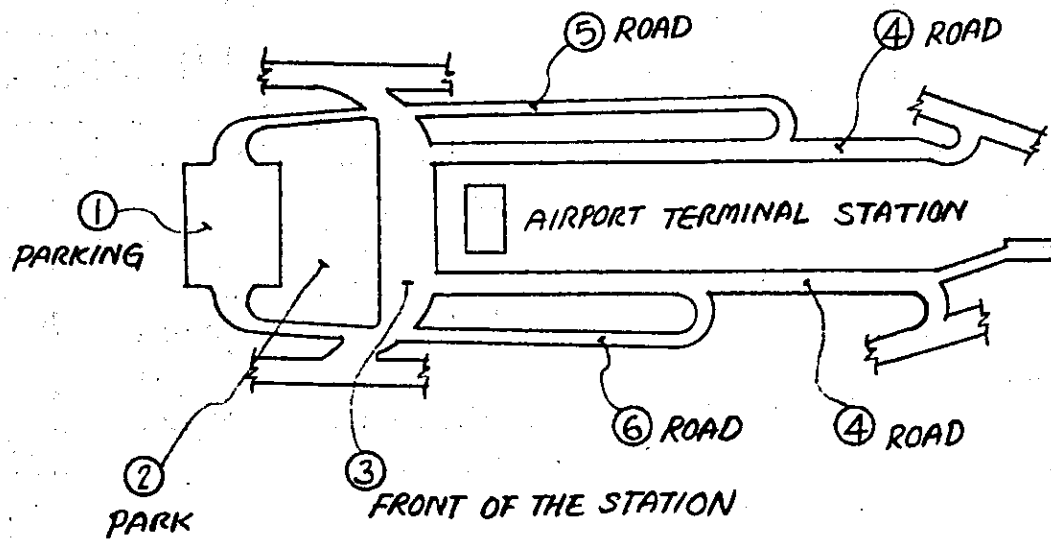
LIGHTING AREA	LUX	X	Y	FIXTURE NO.	NO. (Calc)	NO.	LUMEN (Calc)	REMARKS
(1) SIGNAL CABIN (TYPE-1)							0	
Signal device room	200	8.0	4.0	F2, FL40x1	9.5	10	211	
Power room	200	3.7	2.7	F2, FL40x1	3.9	4	205	
Lavatory	100	2.5	2.7	F1, FL40x1	1.1	1	90	Water proof
Signal control room	300	6.8	5.4	F3, FL40x2	5.6	6	319	
Rest room	150	3.0	2.4	F3, FL40x2	0.7	1	200	
Pantry	150	2.4	2.0	F1, FL40x1	1.2	1	127	
Dressing room	150	2.4	2.0	F1, FL40x1	1.0	1	150	
Inspection room	300	6.0	5.9	F3, FL40x2	5.4	6	331	
Corridor (1FL)	100	1.0	7.0	I3, IL100x1	2.3	3	130	
Corridor (2FL)	100	1.0	5.0	I3, IL100x1	1.7	2	121	

NEW RAILWAY FOR CENGKARENG AIRPORT / INDONESIA

LIGHTING AREA	LUK	W	Y	FIXTURE NO.	NO. (Calc)	NO.	LUK (Calc)	REMARKS
[4] SIGNAL CABIN (TYPE-2)								
Relay equipment room	200	6.9	3.9	F2, FL40#1	8.9	10	225	
Lavatory	100	1.7	0.9	F1, FL40#1	0.3	1	395	Water proof
Signal handling room	300	3.9	3.9	F3, FL40#2	2.5	3	354	
Rest room	150	2.9	3.9	F3, FL40#2	1.2	1	128	

LIGHTING AREA		LUX	X	Y	FIXTURE NO.	HD. (Calc)	ND.	LUX (Calc)	REMARKS
[5] KOTA INTAN STATION									
Concourse	200	9.5	11.0		M1, ML300#1	3.7	4	218	
Passenger waiting & cafe	150	5.5	5.0		I3, IL100#1	8.5	9	160	Water proof
Passengers lavatory (M)	100	4.6	4.8		F2, FL40#1	2.5	4	162	Water proof
Passengers lavatory (M)	100	4.9	4.8		F2, FL40#1	2.6	4	153	
Porter	200	5.0	4.7		F3, FL40#2	2.4	3	249	
Maintenance & janitors	300	8.5	4.8		F4, FL40#2	5.8	6	310	
Station office	300	5.0	11.0		M1, ML300#1	3.0	4	394	
	300	5.0	5.0		F3, FL40#2	3.8	4	312	
Ticket check	300	4.7	4.8		F3, FL40#2	3.5	4	346	
Station master room	300	4.8	5.0		F3, FL40#2	3.7	4	325	
Meeting room	300	4.8	3.6		F3, FL40#2	2.9	3	311	
Security	100	5.0	4.0		F2, FL40#1	2.5	3	120	Water proof
Staff lavatory	200	5.0	4.0		F3, FL40#2	2.5	3	240	
Staff locker room	50	4.6	4.0		F2, FL40#1	1.5	2	66	
Storage 1	50	5.0	3.8		F2, FL40#1	1.2	2	84	
Storage 2	200	5.4	4.1		F4, FL40#2	3.7	4	218	
Power room	200	5.0	3.8		F4, FL40#2	2.4	3	252	
Mechanical room	200	7.4	6.0		F4, FL40#2	4.6	6	264	
Signal telecom device Rm.	50	3.0	6.0		F2, FL40#1	1.5	2	67	
Storage 3	100	6.0	40.0		F5, FL110#1	10.3	12	116	
Corridor 1	100	2.5	40.0		F6, FL110#1	6.5	7	108	
Corridor 2	200	7.0	5.0		F4, FL40#2	4.1	4	196	Water proof
Coverd service area 1	200	7.0	5.0		F4, FL40#2	4.1	4	196	Water proof
Coverd service area 2	200	5.0	7.0		F3, FL40#2	3.3	4	241	
Dining room	300	3.5	3.0		F3, FL40#2	2.2	2	275	
Kitchen	50	1.8	50.0		M1, NL150#1	2.5	4	79	Water proof
Corridor (out door)	100	4.6	2.9		F1, FL40#1	2.4	3	127	Water proof
Ablution room	100	7.7	4.6		I4, IL40#5	7.5	6	79	
Musholla	150	6.9	5.0		F3, FL40#2	2.5	3	183	
Rest room	100	2.9	3.5		F3, FL40#2	0.8	1	119	Water proof
Lavatory	100	1.3	3.0		I3, IL100#1	1.3	2	155	
Corridor	100	6.5	90.0		F6, FL110#1	31.8	50	157	Water proof
Platform									
Train control office	300	4.4	3.4		F3, FL40#2	2.5	3	360	Water proof
Signal telecom office	300	4.4	3.4		F3, FL40#2	2.5	3	360	Water proof
						0.0		0	
Shops	100	30.0	4.3		F4, FL40#2	12.2	12	98	Water proof
Loading yard	100	30.0	4.3		F4, FL40#2	12.2	12	98	Water proof

[6] AIRPORT TERMINAL STATION.
STATION PLAZA



①. PARKING

10 LUX. 50M x 50M HL 250W 8 Nos.

② PARK

10 LUX 25M x 75M HL 250W x 2 3 Nos.

③ FRONT OF THE STATION

50 LUX 20M x 60M HL 700W x 6 1 No.

④ ROAD

10 LUX 8M x 210M HL 250W 6 Nos.

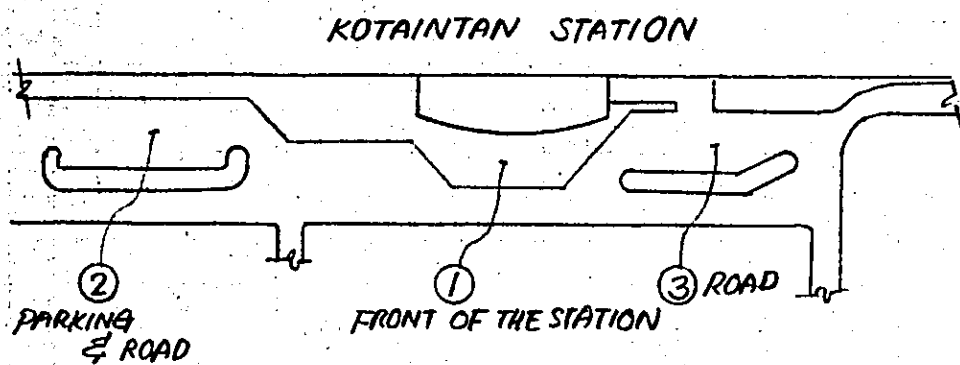
⑤ ROAD

10 LUX 4M x 150M HL 250W 2 Nos.

⑥ ROAD

10 LUX 4M x 130M HL 250W 2 Nos.

[7] KOTAINTAN STATION
STATION PLAZA



① FRONT OF THE STATION

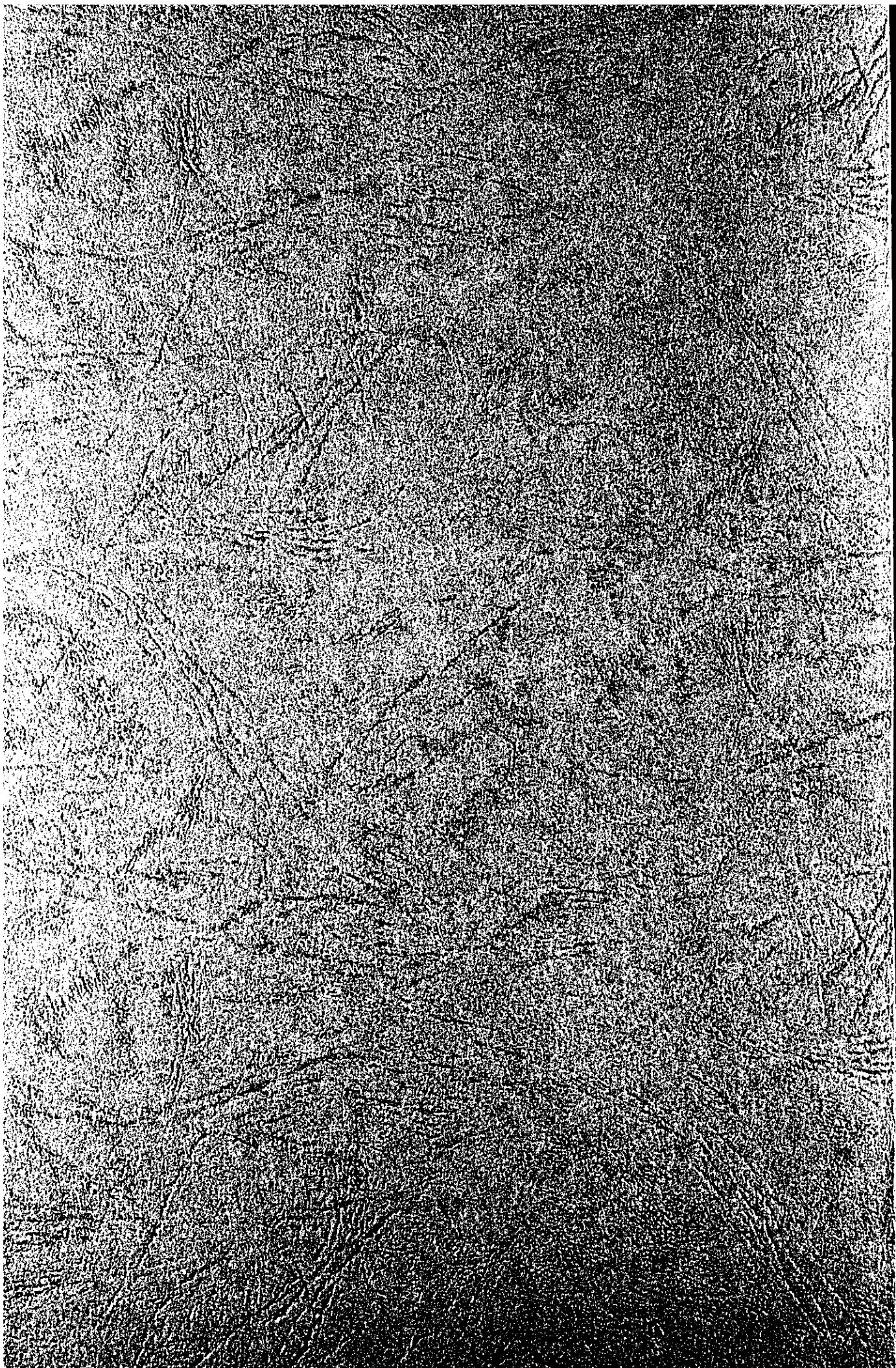
50 LUX 80M x 30M HL 700W x 6 2 Nos.

② PARKING AND ROAD

10 LUX 75M x 35M HL 250W 7 Nos.

③ ROAD

10 LUX 60M x 30M HL 250W 5 Nos.



... ..